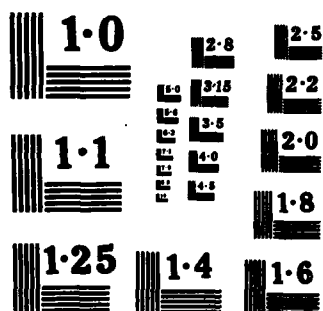


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CENTRE DE RECHERCHES  
DE L'INSTITUT SUPERIEUR INDUSTRIEL CATHOLIQUE  
DU HAINAUT

Stresses and Displacements  
in a four layered System with fixed Bottom

Contract: DAJA-85-C-0013

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Intermediate Report by  
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Intermediate Report.

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(1)

STRESSES AND DISPLACEMENTS IN A FOUR LAYERED  
SYSTEM WITH FIXED BOTTOM

Intermediate report.

Contract: DAJA-85-C-0013 April 30, 1985.

Introduction.

*This document describes,*

The research work necessary to fulfill the requirements of the contract that must lead to the establishment of a computer program able to calculate all stresses and displacements in a four layered system with fixed bottom submitted to a series of loads, is based on:

- existing material: isotropic multilayer theory (BURMISTER, 1943) and anisotropic multilayer theory (VAN CAUWELAERT, 1983);
- original research work: interface conditions (fixed bottom, partial friction) and satisfactory convergency at the surface and in the first layer of the system.

This intermediate report will only deal with the original research work that had to be performed. The required original research is completely terminated, which justifies this report, and at an entirely satisfactory level as will be shown.

This report contains three parts:

1) A complete discussion of the interface conditions.

2) The demonstration that satisfying convergency can be obtained, and in the meantime overflow problems can be eliminated, if the equations are written in closeform although in a sufficient comprehensive form so that the whole problem can still be overlooked but in such a way that all numerical problems can be solved. *and*

3) The complete algebraical analysis leading to the equations in extended form for an isotropic four-layered system. The analysis is also developed for a three-layered system to enable us to compare and to check the results with those obtained by means of reliable existing programs. ←

### 1. The interface conditions.

The stresses and displacements in a layer of a multilayered system are obtained for an isotropic body from following stress function (BURMISTER, 1943):

$$\phi = pa \int_0^{\infty} \frac{J_0(mr) \cdot J_1(ma)}{m} [A_i e^{mz} - B_i e^{-mz} + z C_i e^{mz} - z D_i e^{-mz}] dm$$

and are given by

$$\sigma_z = pa \int_0^{\infty} J_1(mr) \cdot J_1(ma) [A_i m^2 e^{mz} + B_i m^2 e^{-mz} - C_i m(1-2\mu_i - mz) e^{mz} + D_i m(1-2\mu_i + mz) e^{-mz}] dm$$

$$\begin{aligned} \sigma_r = & -pa \int_0^{\infty} J_0(mr) \cdot J_1(ma) [A_i m^2 e^{mz} + B_i m^2 e^{-mz} + C_i m(1+2\mu_i + mz) e^{mz} - D_i m(1+2\mu_i - mz) e^{-mz}] dm \\ & + pa \int_0^{\infty} \frac{J_1(mr) \cdot J_1(ma)}{mr} [A_i m^2 e^{mz} + B_i m^2 e^{-mz} + C_i m(1+mz) e^{mz} - D_i m(1-mz) e^{-mz}] dm \end{aligned}$$

$$\tau_{rz} = -pa \int_0^{\infty} J_1(mr) \cdot J_1(ma) [A_i m^2 e^{mz} - B_i m^2 e^{-mz} + C_i m(2\mu_i + mz) e^{mz} + D_i m(2\mu_i - mz) e^{-mz}] dm$$

$$w = \frac{1+\mu_i}{E_i} pa \int_0^{\infty} \frac{J_0(mr) \cdot J_1(ma)}{m} [A_i m^2 e^{mz} - B_i m^2 e^{-mz} - C_i m(2-4\mu_i - mz) e^{mz} - D_i m(2-4\mu_i + mz) e^{-mz}] dm$$

$$u = -\frac{1+\mu_i}{E_i} pa \int_0^{\infty} \frac{J_1(mr) \cdot J_1(ma)}{m} [A_i m^2 e^{mz} + B_i m^2 e^{-mz} + C_i m(1+mz) e^{mz} - D_i m(1-mz) e^{-mz}] dm$$



where

- $a$  is the radius of a uniformly distributed circular load  
 $p$  is the value of the vertical pressure  
 $r$  is the horizontal distance from the axle in a cylindrical coordinate system  
 $z$  is the depth  
 $\sigma_z$  is the vertical stress  
 $\sigma_r$  is the horizontal radial stress  
 $\tau_{rz}$  is the shearstress  
 $w$  is the vertical deflection  
 $u$  is the radial (horizontal) displacement  
 $E_i$  is the Young modulus of the concerned layer  
 $\mu_i$  is Poisson's ratio of the concerned layer  
 $A_i, D_i$  are unknown parameters to be determined by the boundary conditions  
 $J_0$  is the Besselfunction of the first kind of order zero  
 $J_1$  is the Besselfunction of the first kind of order one  
 $m$  is an integrating parameter

In the case on an anisotropic body they are obtained from (VAN CAUWELAERT, 1983):

$$\phi = pa \int_0^{\infty} \frac{J_0(mr) \cdot J_1(ma)}{m} [A_i e^{mz} - B_i e^{-mz} + C_i e^{ms_i z} - D_i e^{-ms_i z}] dm$$

This stressfunction differs fundamentally from the preceeding one: indeed in putting  $s_i = 1$  in it, we do not obtain the stress function for the isotropic case.

We conclude that the two cases must be handled in a separate way.

The stresses and displacements are given by

$$\sigma_z = pa \int_0^{\infty} J_0(mr) \cdot J_1(ma) \left[ n_i(1+\mu_i) (A_i m^2 e^{mz} + B_i m^2 e^{-mz}) + n_i(n_i+\mu_i) (C_i s_i m^2 e^{s_i m z} + D_i s_i m^2 e^{-s_i m z}) \right] dm$$

$$\sigma_r = -pa \int_0^{\infty} J_0(mr) \cdot J_1(ma) \left[ n_i(1+\mu_i) (A_i m^2 e^{mz} + B_i m^2 e^{-mz}) + \frac{n_i(n_i-\mu_i)}{n_i-\mu_i} (C_i s_i m^2 e^{s_i m z} + D_i s_i m^2 e^{-s_i m z}) \right] dm$$

$$+ pa \int_0^{\infty} \frac{J_1(mr) \cdot J_1(ma)}{mr} n_i(1+\mu_i) [A_i m^2 e^{mz} + B_i m^2 e^{-mz} + C_i s_i m^2 e^{s_i m z} + D_i s_i m^2 e^{-s_i m z}] dm$$

$$\tau_{rz} = -pa \int_0^{\infty} J_1(mr) \cdot J_1(ma) \left[ n_i(1+\mu_i) (A_i m^2 e^{mz} - B_i m^2 e^{-mz}) + n_i(n_i+\mu_i) (C_i s_i m^2 e^{s_i m z} - D_i s_i m^2 e^{-s_i m z}) \right] dm$$

$$w = \frac{1+\mu_i}{E_i} p a \int_0^a \frac{J_0(mr) \cdot J_1(ma)}{m} \left[ n_i (1+\mu_i) (A_i m^2 e^{m^2} - B_i m^2 e^{-m^2}) + \frac{n_i s_i (n_i + \mu_i)^2}{(1+\mu_i)} (C_i s_i m^2 e^{s_i m^2} - D_i s_i m^2 e^{-s_i m^2}) \right] dm$$

$$u = \frac{(1+\mu_i) n_i (n_i + \mu_i)}{E_i} p a \int_0^a \frac{J_1(mr) J_1(ma)}{m} \left[ A_i m^2 e^{m^2} + B_i m^2 e^{-m^2} + C_i s_i m^2 e^{s_i m^2} + D_i s_i m^2 e^{-s_i m^2} \right] dm$$

where

$n_i = E_{vi}/E_{hi}$  is the degree of anisotropy, the ratio between the vertical and the horizontal Young modulus of the concerned layer  
 $\mu_i$  is Poisson's ratio expressing a strain in the horizontal plane induced by a stress in the vertical direction

$s_i = \sqrt{\frac{n_i - \mu_i^2}{n_i^2 - \mu_i^2}}$  is the index of anisotropy.

### 1.1. The partial friction condition.

Let us consider a n-layered system, consisting in (n - 1) layers of a finite thickness built on a semi-infinite body.

For each layer exists a stress function  $\phi_i(A_i, B_i, C_i, D_i)$  with 4 unknown parameters: the total of unknown parameters is 4n.

Two parameters depend on the shape of the load at the surface

$$\sigma_z = f(p) \quad \text{for } r \leq a$$

$$\tau_{rz} = 0$$

At infinite depth stresses and displacements must vanish and thus  $A_n$  and  $C_n = 0$ . We remain with  $4n - 4 = 4(n - 1)$  parameters to be determined with 4 conditions at each interface.

The hypothesis is introduced at this stage that under effect of the load, the layers remain individually fully in contact, which is expressed by imposing that at the bottom of each layer and at the surface of next layer vertical stresses ( $\sigma_z$ ), shear stresses ( $\tau_{rz}$ ) and vertical displacements ( $w$ ) are identic.

The fourth interface condition depends on the relative adhesion in the horizontal plane between the considered layers.

The two extremes are

- full continuity, expressed by setting that the horizontal displacements ( $u$ ) are identic;

- frictionless interface, by considering the interface as a principal plane and thus by setting the shearstresses equal zero.

Partial adhesion has been tentatively introduced by several authors, utilizing, in the same way as WESTERGAARD (1926), a relation between horizontal displacements and shearstress:

$$K(u_i - u_{i+1}) = \tau_{rz_i}$$

where  $u_i$  is the horizontal displacement at the bottom of the  $i$ -th layer and  $u_{i+1}$  that at the surface of the  $(i+1)$ -th layer.

We shall prove that such a relation cannot be correct in the case of a multilayer.

One has, for an isotropic body, following relations between displacements, shearstrains and shearstresses:

$$\begin{aligned} \frac{\partial u_i}{\partial z} + \frac{\partial w}{\partial r} &= \gamma_{rz_i} = [2(1+\mu_i)/E_i] \cdot \tau_{rz_i} \\ \frac{\partial u_{i+1}}{\partial z} + \frac{\partial w_{i+1}}{\partial r} &= \gamma_{rz_{i+1}} = [2(1+\mu_{i+1})/E_{i+1}] \cdot \tau_{rz_{i+1}} \end{aligned}$$

We know from the boundary conditions that

$$w_i = w_{i+1} \quad \tau_{rz_i} = \tau_{rz_{i+1}}$$

This is true everywhere on the interface so that we also can write that

$$\frac{\partial w_i}{\partial r} = \frac{\partial w_{i+1}}{\partial r}$$

By subtraction we obtain

$$\frac{\partial u_i}{\partial z} - \frac{\partial u_{i+1}}{\partial z} = 2 \left[ \frac{1+\mu_i}{E_i} - \frac{1+\mu_{i+1}}{E_{i+1}} \right] \tau_{rz_i} = k' \tau_{rz_i}$$

We can thus write

$$\frac{1}{k'} \left[ \frac{\partial u_i}{\partial z} - \frac{\partial u_{i+1}}{\partial z} \right] = k (u_i - u_{i+1})$$

This relation must be satisfied for all values of the parameter  $m$ , so that one must necessarily have that

$$\frac{k}{k'} \frac{\partial u_i}{\partial z} = u_i \quad \frac{k}{k'} \frac{\partial u_{i+1}}{\partial z} = u_{i+1}$$

The solutions of those differential equations are

$$u_i = e^{2k'/k} \cdot f_1(r) \quad u_{i+1} = e^{2k'/k} \cdot f_2(r)$$

Comparing those solutions with the relation above for the horizontal displacements we conclude that the obtained expressions must be deduced from a stressfunction different from the original one which is nevertheless the unique solution of the

compatibility equations. Compatibility is thus not respected and relation  $k(u_i - u_{i+1}) = \tau_{rz}$  cannot be accepted.

Our meaning is that the only way to express partial continuity (or partial adhesion) consists in writing

$$u_i = k \cdot u_{i+1}$$

with  $k \in [0, \infty]$ .

When  $k = 1$ , one has full continuity

When  $k \neq 1$ , one has partial continuity.

It is necessary now to give a physical sense to the parameter  $k$ .

Excepted the extreme case of a frictionless interface, there will always be some friction between the layers at their interface.

We rely then our approach on Coulomb's definition of friction. If  $\varphi$  is the angle of friction at the interface, there will be no sliding (in the geotechnical sense of the word) between the two layers as long as  $\tau_{rz} < \sigma_z \tan \varphi$ .

The limit value for  $k$ , at which sliding due to shear failure will occur, is then given by  $\tau_{rz}/\sigma_z = \tan \varphi$ .

Beyond this value of  $k$ , shear stresses vanish and the interface has to be considered as frictionless.

We must of course take into account here the stresses due to the wheel load but also those due to the own weight of the layers above the interface, which reduce to vertical stresses only.

The final relation becomes then

$$\frac{\tau_{rz}(\text{traffic})}{\sigma_z(\text{traffic}) + \sigma_z(\text{mass})} = \tan \varphi.$$

The values of the stresses due to traffic varie with the distance to the axle of the load.

We carry then the calculations out in two steps:

- we calculate for different values of  $k$  the maximum value (function from the distance from the axle of the load) of the ratio  $\tau/\sigma$ .
- we determine the limit value of  $k$  in function of  $\tan \varphi$ .

As an illustration of the method, we have taken the most simple case, that of an isotropic two-layer system, with  $H$  the thickness of the first layer.

For simplicity we take  $\mu_1 = \mu_2 = 0.5$

We write  $A_1 m^2 = A_i$ ,  $B_1 m^2 = B_i$ ,  $C_1 m = C_i$ ,  $D_1 m = D_i$

$$F = \frac{E_1(1+\mu_2)}{E_2(1+\mu_1)} = \frac{E_1}{E_2}$$

$$L = k \frac{E_1(1+\mu_1)}{E_2(1+\mu_1)} = k \frac{E_1}{E_2}$$

The boundary conditions are then:

At the surface ( $z = 0$ ):

$$\sigma_z = p \quad A_1 + B_1 = 1$$

$$\tau_{rz} = 0 \quad A_1 - B_1 + C_1 + D_1 = 0$$

At the interface ( $z = H$ ):

$$\sigma_{z_1} = \sigma_{z_2} \quad A_1 e^{mH} + B_1 e^{-mH} + mH C_1 e^{mH} + mH D_1 e^{-mH} \\ = B_2 e^{-mH} + mH D_2 e^{-mH}$$

$$\tau_{rz_1} = \tau_{rz_2} \quad A_1 e^{mH} - B_1 e^{-mH} + (1+mH)C_1 e^{mH} + (1-mH)D_1 e^{-mH} \\ = -B_2 e^{-mH} + (1-mH)D_2 e^{-mH}$$

$$w_1 = w_2 \quad A_1 e^{mH} - B_1 e^{-mH} + mH C_1 e^{mH} - mH D_1 e^{-mH} \\ = F [-B_2 e^{-mH} - mH D_2 e^{-mH}]$$

$$u_1 = k u_2 \quad A_1 e^{mH} + B_1 e^{-mH} + (1+mH)C_1 e^{mH} - (1-mH)D_1 e^{-mH} \\ = L [B_2 e^{-mH} - (1-mH)D_2 e^{-mH}]$$

Solving the system of 6 equations, one obtains at the interface ( $z = H$ ):

$$\sigma_z = 2p\alpha \int_0^\infty J_0(mr) J_1(ma) \left[ (1-mH)(1-L)e^{-3mH} + (1+mH)(1+L)e^{-mH} \right] \frac{dm}{\nabla}$$

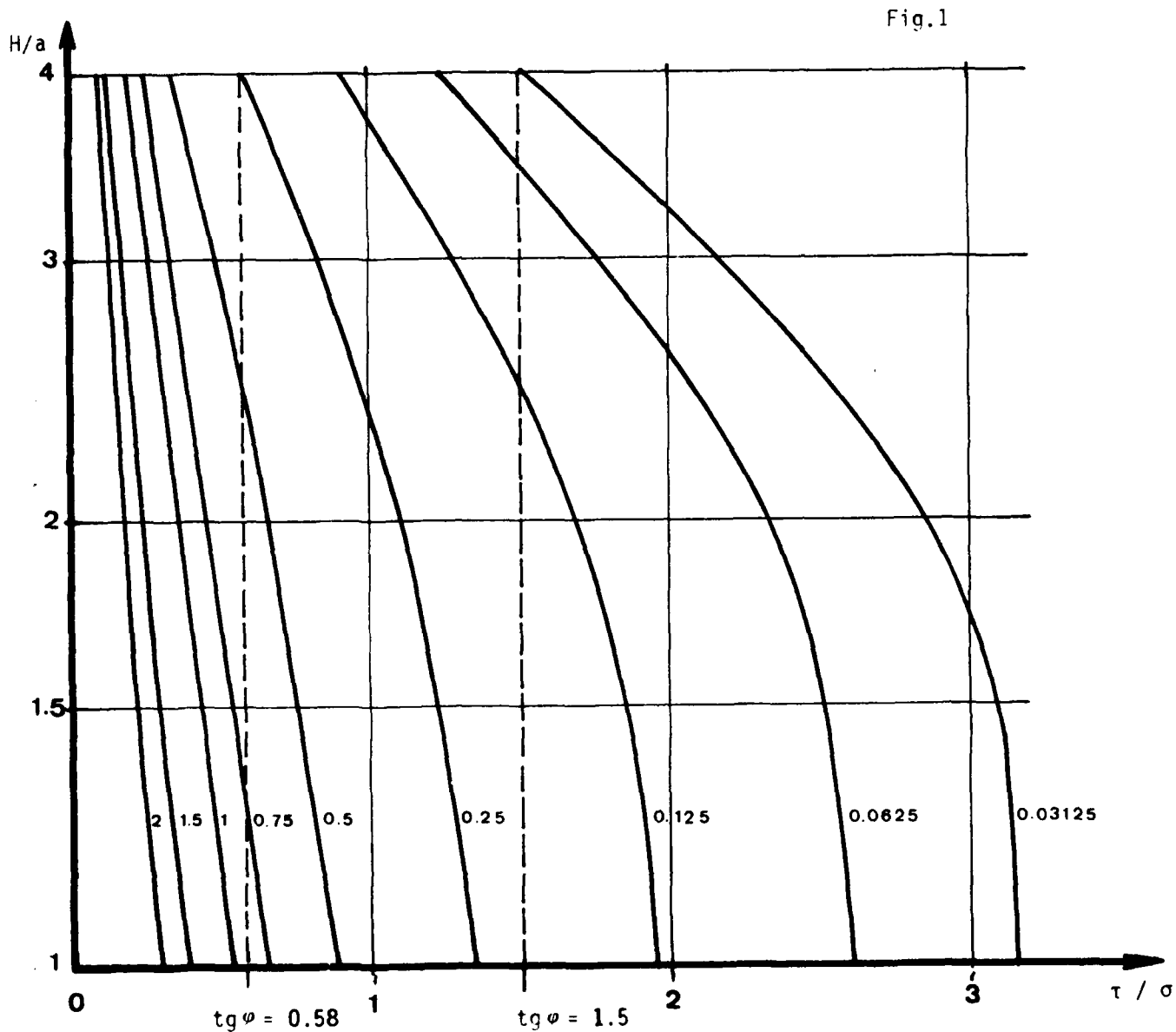
$$\tau_{rz} = 2p\alpha \int_0^\infty J_1(mr) J_1(ma) \left[ (1-F)e^{-3mH} + (1+F)e^{-mH} \right] \frac{dm}{\nabla}$$

$$\nabla = (1-F)(1-L)e^{-4mH} + 2 \left[ (1-FL) - 2mH(F-L) + 2m^2 H^2 (1-FL) \right] e^{-2mH} \\ + (1+F)(1+L)$$

We have performed the computations for different values of  $H/a$ , with  $a = 10$  cm,  $p = 0,6$  MN/m<sup>2</sup> and  $E_1/E_2 = 10$ . We have considered a specific weight of the first layer of 22 kN/m<sup>3</sup>.

The results are given on figure 1: in abscissa one finds the values of the ratio  $\tau/\sigma$  and in ordinate the relative thicknesses.

The curves give for different values of  $k$  the maximum value of  $\tau/\sigma$ .



This is practically impossible.

One could of course be tempted to interrupt the integration procedure when the first integral has converged to a satisfactory level, "hoping" that the second integral can be neglected at that moment.

To illustrate the danger of such an approach we return to the semi-infinite body.

In the case on an isotropic body, the deflection at the surface and the vertical stress at a depth  $z$  are given in the axle of the load by

$$w = -pa \frac{2(1-\mu^2)}{E} \int_0^{\infty} \frac{J_1(ma)}{m} dm \quad (2)$$

$$\sigma_z = pa \int_0^{\infty} J_1(ma) (1-mz) e^{-mz} dm \quad (3)$$

Those integrals can of course be solved analytically

$$w = -pa \frac{2(1-\mu^2)}{E} \quad (4)$$

$$\sigma_z = p \left[ 1 - \frac{z^3}{(a^2+z^2)^{3/2}} \right] \quad (5)$$

We can compare (2) with the second integral of (1) and (3) with the first integral of (1).

We perform then a numerical integration of (2) and (3) and stop the procedure when (3) has converged to a satisfactory level, which is easily checked by comparing the obtained result with the correct one given by (5).

The difference between the numerical result for  $w$  obtained at that moment by integration of (2) with the analytical result given by (4) will give us an illustration of the possible error when integrating (1) and stopping the process when its first integral has converged.

This difference is illustrated in figure 2.

In absciss, we have the convergency level adopted for the vertical stress and in ordinate the error, expressed in %, on the values of the vertical stress and the vertical deflection.

One sees that for even such low levels as  $10^{-3}$ , the value of the vertical stress is absolutely correct, while the error on the deflection varies between - 5% and + 8%, depending on the chosen convergency level and the relative depth at which the vertical stress is computed; worst of all is that we have no means to predict either the direction either the amplitude of the error on  $w$ .

### 2.3. The vertical deflection at the surface.

When one computes the deflections at the surface, convergency is obtained only very slowly.

To illustrate this let us look at the expression of the deflection at the surface developed in § 2.1.

$$w = -pa \frac{2(1-\mu_1^2)}{E_1} \int_0^\infty \frac{J_0(mr) \cdot J_1(ma)}{m} \cdot \left[ \frac{Fe^{2mH} - (2F-1-2mH) - (1-F)e^{-2mH}}{Fe^{2mH} + (2F-1)2mH - (1+2m^2H^2) + (1-F)e^{-2mH}} \right] dm$$

To avoid overflow problems we divide numerator and denominator by  $e^{2mH}$

$$w = -pa \frac{2(1-\mu_1^2)}{E_1} \int_0^\infty \frac{J_0(mr) \cdot J_1(ma)}{m} \cdot \left\{ \frac{F - (2F-1-2mH)e^{-2mH} - (1-F)e^{-4mH}}{F + [(2F-1)2mH - (1+2m^2H^2)]e^{-2mH} + (1-F)e^{-4mH}} \right\} dm$$

For large values of  $m$ , numerator and denominator tend both to  $F$ , so that for, let us say  $m = m_L$ , the expression above could be written as follows:

$$w = -pa \frac{2(1-\mu_1^2)}{E_1} \int_0^{m_L} \frac{J_0(mr) \cdot J_1(ma)}{m} \cdot \left\{ \frac{F - (2F-1-2mH)e^{-2mH} - (1-F)e^{-4mH}}{F + [(2F-1)2mH - (1+2m^2H^2)]e^{-2mH} + (1-F)e^{-4mH}} \right\} dm - pa \frac{2(1-\mu_1^2)}{E_1} \int_{m_L}^\infty \frac{J_0(mr) \cdot J_1(ma)}{m} dm \quad (1)$$

The first integral converges fast, the second converges proportionally to  $1/m$ . This means that if one should want a result correct at  $10^{-5}$ , one has to perform the numerical integration of the second integral until values of  $m$  above 100,000!



Underflow will obviously occur now, but most of the computers have a routine that sets variables subjected to underflow equal to zero. If such a routine does not exist, it is very easy to build it into the program.

But more interesting is the fact that, having transformed the relations for  $C_1$  and  $D_1$ , convergency will occur quite quickly and in a complete safe way: the numerators both tend to zero, while the denominator tends to a constant  $F$ .

This can be obtained automatically in writing the boundary conditions at the surface ( $z = -H$ ) as follows:

$$A_1 e^{-3mH} + B_1 e^{-mH} - C_1 (1 - 2\mu_1 + mH) e^{-3mH} + D_1 (1 - 2\mu_1 - mH) e^{-mH} = e^{-2mH}$$

$$A_1 e^{-3mH} - B_1 e^{-mH} + C_1 (2\mu_1 - mH) e^{-3mH} + D_1 (2\mu_1 + mH) e^{-mH} = 0$$

However this is only true in the case of a two-layer system.

In a three-layer with  $H_1$ , the thickness of the first layer, and  $H_2$ , the thickness of the second layer, occur exponents such as

$$e^{2mH_1 \cdot 2mH_2} \quad \text{and} \quad e^{-2mH_1 \cdot 2mH_2}$$

But they eliminate when writing the denominator in closeform so that dividing the expressions by the largest out of  $e^{2mH_1}$  and  $e^{2mH_2}$  is enough. If one should divide by  $e^{2mH_1 \cdot 2mH_2}$ , the denominator would also tend to zero, which should stop the program because of dividing by zero.

Here is another reason for writing the equations in closeform for three- and more-layers.

## 2.2. Over and underflow problems.

During the integration procedure  $m$  varies from 0 to a value high enough to ensure convergency. We mean by this that the integration procedure can be stopped from the moment on that the terms of the series become so small that they have no more influence on the final result and can thus be neglected.

Practically, however this means that  $m$  can reach quite high values such as 20 or 30 for example.

To illustrate the influence of this, let us go back to the two-layer developed in the preceeding paragraph.

The values of  $C_1$  and  $D_1$ , from which the values of all the other parameters can be deduced, are

$$C_1 = \frac{[(1-F+mH)e^{mH} - (1-F)e^{-mH}]}{Fe^{2mH} + (2F-1).2mH - (1+2m^2H^2) + (1-F)e^{-2mH}}$$

$$D_1 = \frac{[Fe^{mH} - (F-mH)e^{-mH}]}{Fe^{2mH} + (2F-1).2mH - (1+2m^2H^2) + (1-F)e^{-2mH}}$$

The geometrical unities are generally expressed in function of  $a$ , the radius of the load.

Let us consider  $H/a = 5$ .

One immediately sees that no computer can handle exponents as  $e^{mH/a}$  and  $e^{2mH/a}$  without overflow occuring for values of  $m$  above 10.

However this problem can easily be solved by dividing both numerator and denominator by  $e^{2mH}$ :

$$C_1 = \frac{[(1-F+mH)e^{-mH} - (1-F)e^{-3mH}]}{F + [(2F-1).2mH - (1+2m^2H^2)]e^{-2mH} + (1-F)e^{-4mH}}$$

$$D_1 = \frac{[Fe^{-mH} - (F-mH)e^{-3mH}]}{F + [(2F-1).2mH - (1+2m^2H^2)]e^{-2mH} + (1-F)e^{-4mH}}$$

Replacing  $A_1$ ,  $B_1$ ,  $C_1$  and  $D_1$  by their values, the deflection becomes

$$w = -pa \frac{2(1-\mu^2)}{E_1} \int_0^a \frac{J_0(mr) J_1(ma)}{m} \left[ \frac{F e^{2mH} - (2F-1-2mH) - (1-F) e^{-2mH}}{F e^{2mH} + (2F-1)2mH - (1+2m^2 H^2) + (1-F) e^{-2mH}} \right] dm$$

At the origin of the integration ( $m = 0$ ), the term in between brackets becomes indefinite:  $0/0$ .

This has no influence when computing stresses, because the Bessel functions products occurring here are also zero at the origin:  $J_0(mr) \cdot J_1(ma) = 0$  for  $m = 0$ .

But in the case of the deflection

$$\lim_{m \rightarrow 0} \frac{J_0(mr) \cdot J_1(ma)}{m} = \frac{a}{2}$$

It is therefore absolutely necessary to have the term in between brackets in close form to be able to determine its value for  $m = 0$ .

The importance of the first term of the series is not negligible: for  $m = 0$  the term in between brackets is equal to  $E_1 \cdot (1-\mu^2) / [E_2 \cdot (1-\mu^2)]$ .

If  $h$  is the interval chosen for the numerical integration, one can then write

$$w = -pa \frac{2(1-\mu^2)}{E_1} \left\{ \frac{1}{2} \cdot \frac{E_1(1-\mu^2)}{E_2(1-\mu^2)} \cdot \frac{h}{3} + \int_h^a \frac{J_0(mr) J_1(ma)}{m} [ ] dm \right\}$$

and, if we make a semi-infinite body from the two-layer ( $E_1 = E_2$ ,  $\mu_1 = \mu_2$ ):

$$w = -pa \frac{2(1-\mu^2)}{E} \left\{ \frac{h}{6} + \int_h^a \frac{J_0(mr) \cdot J_1(ma)}{m} [ ] dm \right\}$$

Comparing this expression with that for the deflection at the surface of a semi-infinite body

$$w_{\infty} = -pa \cdot 2(1-\mu^2) / E$$

one concludes that the contribution of the first term  $h/6$  is indeed not negligible, especially when we have in mind that the only practical measurement that can be performed on a real roadstructure is the vertical deflection at the surface.

At the surface ( $z = -H$ ):

$$A_1 e^{-mH} + B_1 e^{mH} - C_1 (1 - 2\mu_1 + mH) e^{-mH} + D_1 (1 - 2\mu_1 - mH) e^{mH} = 1$$

$$A_1 e^{-mH} - B_1 e^{mH} + C_1 (2\mu_1 - mH) e^{-mH} + D_1 (2\mu_1 + mH) e^{mH} = 0$$

At the frictionless interface ( $z = 0$ ):

$$A_1 + B_1 - C_1 (1 - 2\mu_1) + D_1 (1 - 2\mu_1) = B_2 + D_2 (1 - 2\mu_2)$$

$$A_1 - B_1 + 2\mu_1 C_1 + 2\mu_1 D_1 = 0$$

$$-B_2 + 2\mu_2 D_2 = 0$$

$$\begin{aligned} \frac{1+\mu_1}{E_1} [A_1 - B_1 - 2C_1 (1 - 2\mu_1) - 2D_1 (1 - 2\mu_1)] \\ = \frac{1+\mu_2}{E_2} [-B_2 - 2D_2 (1 - 2\mu_2)] \end{aligned}$$

Solving the system for  $C_1$  and  $D_1$ , one obtains (BURMISTER, 1943)

$$C_1 = [(1 - F + mH) e^{mH} - (1 - F) e^{-mH}] \cdot \frac{1}{\nabla}$$

$$D_1 = [F e^{mH} - (F - mH) e^{-mH}] \cdot \frac{1}{\nabla}$$

where

$$\nabla = F e^{2mH} + (2F - 1) \cdot 2mH - (1 + 2m^2 H^2) + (1 - F) e^{-2mH}$$

$$F = \frac{(1 - \mu_2) + n(1 - \mu_1)}{2(1 - \mu_2)} \quad n = \frac{E_2}{E_1} \cdot \frac{(1 + \mu_1)}{(1 + \mu_2)}$$

$$A_1 = C_1 (F - 2\mu_1) - D_1 (1 - F)$$

$$B_1 = C_1 F - D_1 (1 - 2\mu_1 - F)$$

The vertical deflection at the surface is

$$\begin{aligned} w = p a \frac{1+\mu_1}{E_1} \int_0^\infty \frac{J_0(mr) J_1(ma)}{m} [A_1 m^2 e^{-mH} - B_1 m^2 e^{mH} \\ - (2 - 4\mu_1 + mH) C_1 m e^{-mH} - (2 - 4\mu_1 - mH) D_1 m e^{mH}] dm \end{aligned}$$

## 2. Solution of particular numerical problems (convergency-problems).

### 2.1 The full slip interface condition.

The value of any stress or displacement is obtained from one of the above mentioned relations.

Let us consider, for example, the vertical stress in the  $i$ -th layer of an anisotropic layer:

$$\sigma_z = pa \int_0^{\infty} J_0(mr) \cdot J_1(ma) \left[ n_i(1+\mu_i)(A_i m^2 e^{mz} + B_i m^2 e^{-mz}) + n_i(n_i+\mu_i)(C_i s_i m^2 e^{s_i m z} + D_i s_i m^2 e^{-s_i m z}) \right] dm$$

The integration can only be performed numerically.

Thus one must calculate the value of the stress for a set of values of  $m$  growing from 0 to a value high enough to ensure convergency.

For each value of  $m$ , those of the parameters  $A_i$ ,  $B_i$ ,  $C_i$  and  $D_i$  must be determined out of the set of boundary conditions, a system of  $(4n - 1)$  equations with  $(4n - 1)$  unknowns in the case of a fixed bottom and  $n$  layers above it.

The first programs solved this problem by inverting the matrix of the  $(4n - 1)$  unknowns. Nevertheless the inversion procedure leads in some cases to unsolvable difficulties because of the presence of the negative exponents tending to zero in the determinant of the denominator.

Other programs have tried to avoid the inversion procedure as follows: one chooses appropriate values for  $B_n$  and  $D_n$ , goes through the whole set of equations and verifies in how far the surface conditions are met. One then chooses another pair of values for  $B_n$  and  $D_n$  and follows the same procedure. Since the whole process is linear, the correct values for  $B_n$  and  $D_n$  can finally be obtained by linear interpolation after two runs. The difficulty lies in the appropriate choice of the values of  $B_n$  and  $D_n$  to ensure a numerically correct interpolation.

However, even those programs are not entirely appropriate for the cases with frictionless conditions at some interfaces.

We shall show this with the most simple case, that of a two layer.

Writing  $A_i, B_i, C_i, D_i$  instead of  $A_i m^2, B_i m^2, C_i m, D_i m$ , the boundary conditions are in the case of two isotropic layers, with the origin ( $z = 0$ ) at the interface, the thickness of the first layer being  $H$  and the second layer semi-infinite:

We also conclude that the relative influence on the deflection is much more important when we fix the horizontal displacements ( $u = 0$ ). This should be the case in a laboratory testpit with lateral walls, but less in the case of a real road where lateral movements are not restricted.

The relative influence of the condition  $\tau_{rz} = 0$  is also more important than that of the condition  $w = 0$ , although less important than the condition  $u = 0$ . It seems nevertheless very unlikely that there would be no friction between the subground and the last layer.

The easiest way to fix the bottom from a mathematical point of view is on the other hand the condition  $w = 0$ .

Taking then into account the little influence of the chosen condition on the deflection at the surface and the fact that conditions  $u = 0$  and  $\tau_{rz} = 0$  have less physical sense, we shall retain the condition  $w = 0$  as the most indicated fixed bottom condition.

The deflection at the surface is given by the same relation as above with the appropriate value for C.

One sees that in the 3 cases, the deflection is composed of a first term

$$w_{\infty} = pa \frac{s(1-n)}{E(1-s)} \int_0^{\infty} \frac{J_1(am)}{m} dm = pa \frac{s(1-n)}{E(1-s)}$$

This term is the deflection on top of a semi-infinite body.

In the case of an isotropic body, one has ( $n = 1$ ):

$$w_{\infty} = -pa \frac{2(1-\mu^2)}{E}$$

The second term  $w_r$  depends on the chosen boundary condition; but in the three cases it reduces the value of  $w_{\infty}$  because of the fixed bottom.

We have computed the values of  $w_{\infty}$  and  $w_r$  for different values of  $H/a$ . The results are given below, at a factor  $(1+\mu)/E$ , for  $s = 0.5$  and for the isotropic case.

$s = 0.5$        $w_{\infty} = 1.025$

$H/a$	$w_r (\tau_{rz} = 0)$	$w_r (\mu = 0)$	$w_r (w = 0)$
1	0.339	0.729	0.153
2	0.120	0.295	0.043
3	0.058	0.147	0.019
4	0.034	0.086	0.011
5	0.022	0.057	0.007
6	0.015	0.040	0.005
7	0.011	0.029	0.004
8	0.009	0.023	0.003
9	0.007	0.018	0.002
10	0.006	0.015	0.002

$n = 1$        $w_{\infty} = 1.000$

$H/a$	$w_r (\tau_{rz} = 0)$	$w_r (\mu = 0)$	$w_r (w = 0)$
1	0.402	0.934	0.276
2	0.143	0.442	0.086
3	0.069	0.231	0.040
4	0.040	0.138	0.023
5	0.026	0.091	0.015
6	0.018	0.064	0.010
7	0.013	0.048	0.008
8	0.010	0.037	0.006
9	0.008	0.029	0.005
10	0.007	0.024	0.004

We conclude that from a depth of about  $H/a = 5$ , the absolute influence of the fixed boundary is negligible. This influence will still be much lesser in the case of a roadstructure where the E-moduli of the layers are sensitively higher than the modulus of the subground.

We now choose an appropriate function  $f(r)$  so that  $w$  becomes zero for  $z = H$ .

$$f(r) = - \frac{1+\mu}{E} p a \int_0^\infty \frac{J_0(mr) J_1(ma)}{m} \left[ n(1+\mu) (A m^2 e^{mH} - B m^2 e^{-mH}) \right. \\ \left. + \frac{ns(n+\mu)^2}{(1+\mu)} (C s m^2 e^{smH} - D s m^2 e^{-smH}) \right] dm$$

The final expression for the deflection is then

$$w = w_1 + f(r)$$

One verifies that  $f(r)$  is indeed only a function of  $r$  and that  $w = 0$  for  $z = H$ .

For the same reasons as those developed in § 1.1.2, one of the parameters

$A$  or  $C$  must be zero.

The other parameter is obtained by a supplementary boundary condition (a mechanical condition):

- $\tau_{rz} = 0$  at the depth  $H$
- $u = 0$  at the depth  $H$ .

If we still suppose  $s < 1$  and thus  $A = 0$ , one obtains in the case that  $\tau_{rz} = 0$

$$ns(n+\mu) C s m^2 = \frac{e^{-Hm} e^{-sHm} - e^{-2sHm}}{2e^{-Hm} e^{-sHm} - (1-s) - (1+s)e^{-2sHm}}$$

The deflection at the surface is then

$$w = p a \frac{s(1-n)}{E(1-s)} \int_0^\infty \frac{J_1(am)}{m} dm \\ + p a \frac{1+\mu}{E} \int_0^\infty \frac{J_1(am)}{m} \left\{ \frac{s}{(1-s)} \frac{(n+\mu)}{(1+\mu)} e^{-sHm} - \frac{s}{1-s} e^{-mH} \right. \\ \left. + ns(n+\mu) C s m^2 \left[ \frac{2s(m-1)}{(1-s)(1+\mu)} + \frac{2s}{1-s} e^{-mH} \right. \right. \\ \left. \left. - \frac{m+\mu}{1+\mu} s e^{sHm} - \frac{m+\mu}{1+\mu} s \frac{1+s}{1-s} e^{-smH} \right] \right\} dm$$

In the case  $u = 0$ :

$$ns(n+\mu) C s m^2 = \frac{e^{-sHm} [(n+\mu) s e^{-Hm} - (1+\mu) e^{-sHm}]}{2s(n+\mu) e^{-Hm} e^{-sHm} + (1+\mu)(1-s) - (1+\mu)(1+s) e^{-2sHm}}$$



### 1.2.2. Fixed bottom expressed by mechanical condition only.

Referring to the stress and displacements equations given in paragraph 1, the condition  $w = 0$  at a depth  $H$  is written

$$n(1+\mu)(Am^2e^{mH} - Bm^2e^{-mH}) + \frac{ns(n+\mu)^2}{(1+\mu)}(Csm^2e^{smH} - Dsm^2e^{-smH}) = 0$$

Replacing  $B$  and  $D$  by their values

$$\begin{aligned} & Am^2 \left[ n(1+\mu)e^{mH} - \frac{n(1+s)(1+\mu)}{(1-s)}e^{-mH} + \frac{2ns(n+\mu)}{(1-s)}e^{-smH} \right] \\ & + Csm^2 \left[ \frac{ns(n+\mu)^2}{(1+\mu)}e^{smH} - \frac{2ns(n+\mu)}{(1-s)}e^{-mH} + \frac{ns(1+s)(n+\mu)^2}{(1+\mu)(1-s)}e^{-smH} \right] \\ & = \frac{s(n+\mu)}{(1+\mu)} \frac{1}{1-s} e^{-smH} - \frac{s}{1-s} e^{-mH} \end{aligned}$$

During the integration proces, the value of  $m$  tends to infinity, so that the limit expression of the equation becomes

$$\lim_{m \rightarrow \infty} [Am^2(1+\mu)^2e^{mH} + Csm^2s(n+\mu)^2e^{smH}] = 0$$

This relation can only be satisfied by setting

$$C = 0 \quad \text{when} \quad s > 1$$

$$A = 0 \quad \text{when} \quad s < 1$$

It is easily shown that in the case of an isotropic body, one must always have  $C = 0$ , because of the factor  $z$  multiplying  $C$  in the stressfunction.

Taking  $s < 1$ ,  $C$  becomes

$$ns(n+\mu)Csm^2 = \frac{e^{-smH}[(1+\mu)e^{-mH} - (n+\mu)e^{-smH}]}{2(1+\mu)e^{-mH}e^{-smH} - (n+\mu)(1-s) - (1+s)(n+\mu)e^{-2smH}}$$

The expression of the deflection at the surface and in the axle of the load [ $J_0(mr) = 1$  for  $r=0$ ] is

$$w = \frac{1+\mu}{E} \cdot pa \int_0^{\infty} \frac{J_1(ma)}{m} \left[ 1 - 2ns(n+\mu)Csm^2 \right] \frac{s(1-n)}{(1+\mu)(1-s)} dm$$

### 1.2.3. Fixed bottom by a geometrical and a mechanical condition.

The vertical deflection at a depth  $z$  is given by

$$\begin{aligned} w_1 = & \frac{1+\mu}{E} \cdot pa \int_0^{\infty} \frac{J_0(mr)J_1(ma)}{m} \left[ n(1+\mu)(Am^2e^{mz} - Bm^2e^{-mz}) \right. \\ & \left. + \frac{ns(n+\mu)^2}{(1+\mu)}(Csm^2e^{smz} - Dsm^2e^{-smz}) \right] dm \end{aligned}$$

But with the general solutions of the compatibility equations in multilayer theory, we can do it also in another way by expressing that  $w = 0$  at the desired depth and determining the corresponding values of the parameters  $A_n$ ,  $B_n$ ,  $C_n$  and  $D_n$ .

Since there are thus several possibilities to express a same boundary condition, it is necessary to compare the results obtained and retain the one that seems the most appropriate.

To do this we shall consider the most simple case, that of the semi-infinite body: the one-layer case.

### 2.1. Basic equations.

Let us consider a semi-infinite anisotropic body submitted to a uniformly distributed vertical pressure at its surface.

The stressfunction is

$$\phi = pa \int_0^{\infty} \frac{J_0(mr) \cdot J_1(ma)}{m} [Ae^{mz} - B\bar{e}^{mz} + Ce^{msz} - D\bar{e}^{msz}] dm$$

The surface boundary conditions ( $\sigma_z = p$ ,  $\tau_{rz} = 0$ ,  $z = 0$ ) are deduced from the relations given in § 1. for the stresses

$$n(1+\mu)(Am^2 + Bm^2) + n(n+\mu)(Csm^2 + Dsm^2) = 1$$

$$n(1+\mu)(Am^2 - Bm^2) + ns(n+\mu)(Csm^2 - Dsm^2) = 0$$

Solving this system for B and D, one obtains

$$n(1-s)(1+\mu) Bm^2 = -s + n(1+s)(1+\mu) Am^2 + 2ns(n+\mu) Csm^2$$

$$n(1-s)(n+\mu) Dsm^2 = 1 - 2n(1+\mu) Am^2 - n(1+s)(n+\mu) Csm^2$$

The next step depends on the boundary condition that fixes the deflection at a depth H.

### 1.2. The fixed bottom condition.

The boundary conditions discussed in previous alineas implicate that the last layer of the multilayer is considered as a semi-infinite body.

One can also consider the case of a multilayer built on an undeformable body, that thus any vertical displacement vanishes at the contact face with the undeformable body: we shall call this a fixed bottom condition.

This condition can be introduced in several manners and thus demands a detailed analysis.

A vertical displacement is obtained by integration of the vertical strain:

$$w = \int \epsilon_z \cdot dz$$

It would not be correct to resort to an integration between limits, such as

$$w = \int_0^H \epsilon_z \cdot dz$$

where H could, for example, be the depth at which we want the bottom to be fixed.

In doing so, we would ignore the contribution (zero or not) to the vertical deflection (or displacement) due to other parts of the body that we neglect by integrating between specified limits.

The correct way consists in writing (TIMOSHENKO, 1970):

$$w = \int \epsilon_z \cdot dz + f(r)$$

where  $f(r)$  is a function of  $r$  only, and thus a constant regarding  $z$ , so that by differentiating we obtain again

$$\frac{\partial w}{\partial z} = \epsilon_z$$

By choosing an appropriate expression for  $f(r)$ , we can obtain the bottom fixed at the desired depth; by doing this, we introduce in fact a geometrical condition fixing the reference level for the vertical deflections at the chosen depth.

Let us consider an average value of  $\tan \varphi = 1.5$ , value utilized in the design of continuously reinforced concrete pavements (Mc CULLOUGH, 1981).

We deduce from figure 1 the limit values for  $k$ :

$H/a = 1$	$k = 0.20$
$H/a = 2$	$k = 0.17$
$H/a = 3$	$k = 0.10$
$H/a = 4$	$k = 0.03$

If we consider a surface layer built on a sand basecourse ( $\varphi = 30^\circ$ ,  $\tan \varphi = 0.58$ ), the limit values become:

$H/a = 1$	$k = 1$
$H/a = 2$	$k = 0.60$
$H/a = 3$	$k = 0.40$
$H/a = 4$	$k = 0.25$

With our approach, the case of full continuity ( $u_i = u_{i+1}$ , often called full friction) becomes a particular case for which the angle of friction between the layers corresponds with a value of  $k = 1$ . For other values of  $k$  we have partial continuity ( $u_i \neq u_{i+1}$ , what could be called partial friction or partial adhesion).

We notice that for values of  $k$  smaller than one,  $u_i$  is smaller than  $u_{i+1}$ . This means practically that the lateral movements of the surface layer are retained, for example by shoulders. Such a construction reduces the vertical stresses on the subground which improves the lifetime of the road structure. For values of  $k$  larger than one,  $u_i$  is also larger than  $u_{i+1}$ . Here the lateral movements of the surface layer are easier than those of the sublayer, as in overlay constructions for example.

For the limit value  $k = \infty$ , one obtains also  $T_{rz} = 0$ .

This case is called the frictionless interface.

For  $k = \infty$ , one obtains indeed

$$\sigma_z = 2pa \int_0^a J_0(mr) J_1(ma) \frac{[(1+mH)e^{-mH} - (1-mH)e^{-3mH}] \cdot E_2}{(E_1 - E_2)e^{-4mH} + 2[2mHE_2 - E_1 - 2m^2H^2E_1]e^{-2mH} + (E_1 + E_2)} \cdot dm$$

which is the value for  $\sigma_z$  in the case of a frictionless interface (BURMISTER, 1943) as will be shown in § 2.1.

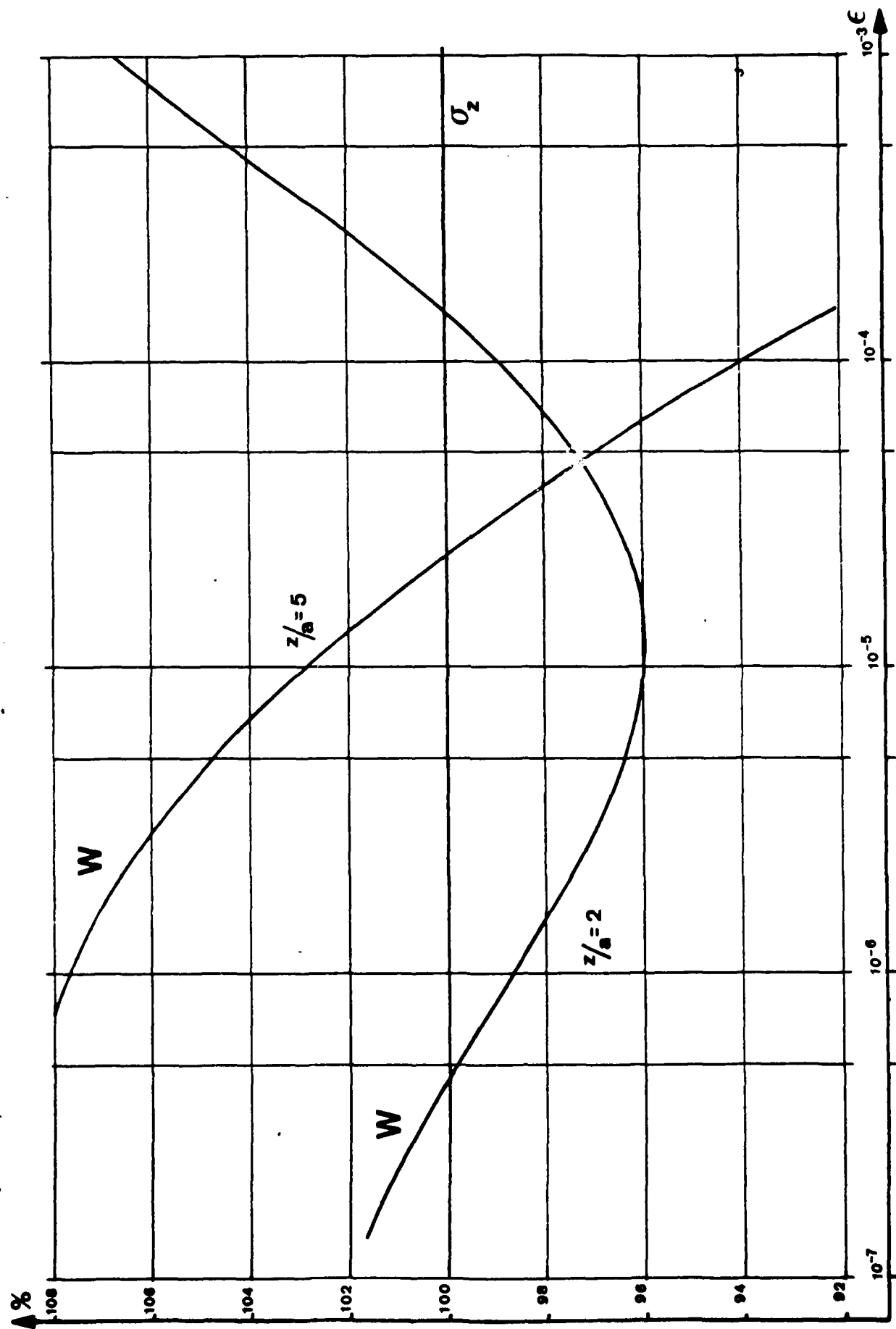


Figure 2

The only way to solve the problem satisfactory is to split the expression of the deflection in another way than the one we had done.

We first write the expression of the deflection with negative exponents only:

$$w = -pa \frac{2(1-\mu^2)}{E_1} \int_0^\infty \frac{J_0(mr) \cdot J_1(ma)}{m} \cdot \left\{ \frac{F - (2F-1-2mH)e^{-2mH} - (1-F)e^{-4mH}}{F + [(2F-1)2mH - (1+2m^2H^2)]e^{-2mH} + (1-F)e^{-4mH}} \right\} dm$$

We then divide the numerator of the term in between brackets by the denominator:

$$w = -pa \frac{2(1-\mu^2)}{E_1} \int_0^\infty \frac{J_0(mr) \cdot J_1(ma)}{m} \cdot \left\{ 1 + 2 \frac{[(1-F)(1+mH) + m^2H^2]e^{-2mH} - (1-F)e^{-4mH}}{F + [(2F-1)2mH - (1+2m^2H^2)]e^{-2mH} + (1-F)e^{-4mH}} \right\} dm$$

and split the integral into two parts from which the first is integrable analytically and the second converges in the usual way.

For  $r = 0$ , one has

$$\int_0^\infty \frac{J_1(ma)}{m} dm = 1$$

For  $r = a$ , one has

$$\int_0^\infty \frac{J_0(ma) \cdot J_1(ma)}{m} dm = 2/\pi$$

For  $r < a$ , one has

$$\int_0^\infty \frac{J_0(mr) \cdot J_1(ma)}{m} dm = F\left(\frac{1}{2}, -\frac{1}{2}; 1; \frac{r^2}{a^2}\right)$$

where  $F$  is the hypergeometric function of GAUSS:

$$F(a, b; c; z) = \sum_{n=0}^{\infty} \frac{(a)_n (b)_n}{n! (c)_n} z^n$$

$$(a)_n = a(a+1)(a+2) \dots (a+n-1) \quad (a)_0 = 1$$

For  $r > a$ , one has

$$\int_0^{\infty} \frac{J_0(mr) \cdot J_1(ma)}{m} dm = \frac{a}{2r} F\left(\frac{1}{2}, \frac{1}{2}; 2; \frac{a^2}{r^2}\right)$$

The obtained result will now be correct, while convergency is reached as fast as for the other equations for stresses.

But again, if we are to be able to compute as indicated, we must have the equations in closeform at our disposal, although in such a form that the integral can be split.

#### 2.4. Convergency in the first layer.

As for the deflection at the surface, the numerical computation of the stresses in the first layer, in fact nearby the surface, also converges very slowly.

To illustrate this let us look at the relation for the vertical stress in the first layer (the equation is given in § 1.):

$$\sigma_z = \rho a \int_0^a J_0(mr) J_1(ma) \left[ A_1 m^2 e^{mz} + B_1 m^2 e^{-mz} - C_1 m(1-2\mu, -mz) e^{mz} + D_1 m(1-2\mu, +mz) e^{-mz} \right] dm$$

We replace  $A_1, B_1, C_1, D_1$  by their values obtained in § 2.1.

$$\sigma_z = \rho a \int_0^a J_0(mr) J_1(ma) \cdot \left\{ F[1+m(H+z)] e^{-mH} \cdot e^{-mz} + \left[ \right] e^{-mH} \cdot e^{mz} + \left[ \right] e^{-3mH} \cdot e^{mz} + \left[ \right] e^{-3mH} \cdot e^{-mz} \right\} \frac{dm}{\Delta}$$

The values of  $z$  are negative in the first layer ( $z = 0$  at the interface).

The term

$$F[1+m(H+z)] e^{-mH} \cdot e^{-mz} \cdot \frac{1}{\Delta}$$

converges very slowly for values of  $-z$  nearly equal to  $H$ , the other terms converge normally.

To solve the problem created by the first term we divide again numerator by denominator:

$$F[1+m(H+z)] e^{-mH} \cdot e^{-mz} \cdot \frac{1}{\Delta} = \frac{[1+m(H+z)] e^{-mH} \cdot e^{-mz}}{F + [(2F-1)2mH - (1+2m^2H^2)] e^{-2mH} + (1-F) e^{-4mH}}$$

The second term of the second member converges normally so that we can again split the integral in several parts from which the one that converges slowly is

$$\rho a \int_0^a J_0(mr) \cdot J_1(ma) [1+m(H+z)] e^{-mH} \cdot e^{-mz} dm.$$



This integral is known as a LIPSCHITZ-HANKEL integral, but only some particular cases are integrable analytically. To solve the problem for all cases we have to make a detour through the analysis of stresses and displacements in a semi-infinite body submitted to an isolated local force  $P$ .

#### 2.4.1. Stresses and displacements under an isolated load.

The Hankel transform in the case of a uniformly distributed load is

$$\psi(m) = pa \int_0^a \frac{J_1(am)}{m} \phi dm$$

We consider the resulting load  $P = \pi pa^2$  acting on a surface whose area reduces to zero.

$$\begin{aligned} \psi(m) &= \lim_{a \rightarrow 0} \frac{P}{\pi} \int_0^a \frac{J_1(am)}{am} \phi dm \\ &= \frac{P}{2\pi} \int_0^\infty \phi dm \end{aligned}$$

The relations for the stresses and the displacements are then given, in the case of an isotropic body, by

$$\begin{aligned} \sigma_r &= \frac{P}{2\pi} \left[ \int_0^\infty m J_0(mr) e^{-mz} dm - 2 \int_0^\infty m^2 J_0(mr) e^{-mz} dm \right. \\ &\quad \left. - (1-2\nu) \int_0^\infty \frac{J_1(mr)}{r} e^{-mz} dm + 2 \int_0^\infty m \frac{J_1(mr)}{r} e^{-mz} dm \right] \\ &= \frac{P}{2\pi} \left\{ \frac{z}{(z^2+r^2)^{3/2}} - \left[ \frac{2z}{(z^2+r^2)^{3/2}} - \frac{3zr^2}{(z^2+r^2)^{5/2}} \right] \right. \\ &\quad \left. - (1-2\nu) \frac{(r^2+z^2)^{1/2} - z}{r^2(r^2+z^2)^{1/2}} + \frac{z}{(z^2+r^2)^{3/2}} \right\} \\ \sigma_z &= \frac{P}{2\pi} \left[ \int_0^\infty m J_0(mr) e^{-mz} dm + 2 \int_0^\infty m^2 J_0(mr) e^{-mz} dm \right] \\ &= \frac{P}{2\pi} \left\{ \frac{z}{(z^2+r^2)^{3/2}} + \left[ \frac{2z}{(z^2+r^2)^{3/2}} - \frac{3zr^2}{(z^2+r^2)^{5/2}} \right] \right\} \end{aligned}$$

$$\tau_{rz} = \frac{Pz}{2\pi} \int_0^{\infty} m^2 J_1(mr) e^{-mz} dm$$

$$= \frac{P}{2\pi} \frac{3z^2 r}{(r^2 + z^2)^{3/2}}$$

$$w = -\frac{P}{2\pi} \frac{2(1-\mu^2)}{E} \int_0^{\infty} J_0(mr) e^{-mz} dm - \frac{Pz}{2\pi} \frac{(1+\mu)}{E} \int_0^{\infty} m J_0(mr) e^{-mz} dm$$

$$= -\frac{P}{2\pi} \frac{2(1-\mu^2)}{E} \frac{1}{(z^2 + r^2)^{1/2}} - \frac{P}{2\pi} \frac{(1+\mu)}{E} \frac{z}{(z^2 + r^2)^{3/2}}$$

$$u = \frac{P}{2\pi} \frac{(1+\mu)(1-2\mu)}{E} \int_0^{\infty} J_1(mr) e^{-mz} dm - \frac{Pz}{2\pi} \frac{(1+\mu)}{E} \int_0^{\infty} m J_1(mr) e^{-mz} dm$$

$$= \frac{P}{2\pi} \frac{(1+\mu)(1-2\mu)}{E} \frac{(r^2 + z^2)^{1/2} - z}{r(r^2 + z^2)^{1/2}} - \frac{P}{2\pi} \frac{(1+\mu)}{E} \frac{zr}{(z^2 + r^2)^{3/2}}$$

#### 2.4.2. Stresses and displacements under a uniformly distributed load.

The stresses and displacements under a uniformly distributed load can be obtained by integrating the relations under an isolated load over the concerned area

$$\sigma_D = \int_0^{2\pi} \int_0^a \sigma_I p dp d\theta$$

where  $\sigma_I$  is given by one of the relations of § 2.4.1. wherein the distance  $r$  must be replaced by  $(r^2 + p^2 - 2rp \cos \theta)^{1/2}$

#### 2.4.3. Relations for the stress $\sigma_r$ .

The relation for  $\sigma_r$  under a distributed load is given by

$$\begin{aligned} \sigma_r = & pa \int_0^{\infty} J_0(mr) \cdot J_1(ma) e^{-mz} dm - pa \int_0^{\infty} J_0(mr) \cdot J_1(ma) m z e^{-mz} dm \\ & - pa (1-2\mu) \int_0^{\infty} \frac{J_1(mr)}{mr} J_1(ma) e^{-mz} dm + pa \int_0^{\infty} \frac{J_1(mr)}{r} J_1(ma) z e^{-mz} dm \end{aligned}$$

so that comparing with the expression for the stress under an isolated load we can conclude that

$$I_1: pa \int_0^a J_0(mr) J_1(ma) e^{-mz} dm = \frac{p}{2\pi} \int_0^{2\pi} \int_0^a \frac{zp}{(x^2+r^2+p^2-2rp \cos \theta)^{3/2}} dp d\theta$$

$$I_2: pa \int_0^a J_0(mr) J_1(ma) mz e^{-mz} dm = \frac{p}{2\pi} \int_0^{2\pi} \int_0^a \left[ \frac{2zp}{(x^2+r^2+p^2-2rp \cos \theta)^{3/2}} - \frac{3x(r^2+p^2-2rp \cos \theta) \cdot p}{(x^2+r^2+p^2-2rp \cos \theta)^{5/2}} \right] dp d\theta$$

$$I_3: pa \int_0^a \frac{J_1(mr)}{mr} J_1(ma) e^{-mz} dm = \frac{p}{2\pi} \int_0^{2\pi} \int_0^a \frac{(x^2+r^2+p^2-2rp \cos \theta)^{1/2} - 2}{(r^2+p^2-2rp \cos \theta)(x^2+r^2+p^2-2rp \cos \theta)^{1/2}} p dp d\theta$$

$$I_4: pa \int_0^a \frac{J_1(mr)}{r} J_1(ma) x e^{-mz} dm = \frac{p}{2\pi} \int_0^{2\pi} \int_0^a \frac{z p dp d\theta}{(x^2+r^2+p^2-2rp \cos \theta)^{3/2}}$$

#### 2.4.4. Relations for the stress $\sigma_z$ .

These relations can be deduced from those established for the stress  $\sigma_r$

#### 2.4.5. Relation for the stress $\tau_{rz}$ .

$$\tau_{rz} = pa \int_0^a J_1(mr) \cdot J_1(ma) m z e^{-mz} dm$$

$$I_5: pa \int_0^a J_1(mr) \cdot J_1(ma) m z e^{-mz} dm = \frac{p}{2\pi} \int_0^{2\pi} \int_0^a \frac{3x^2(r^2+p^2-2rp \cos \theta)^{1/2}}{(x^2+r^2+p^2-2rp \cos \theta)^{5/2}} p dp d\theta$$

#### 2.4.6. Relations for the vertical displacement $w$ .

$$w = -pa \frac{2(1-\mu^2)}{E} \int_0^a \frac{J_0(mr) \cdot J_1(ma)}{m} e^{-mz} dm$$

$$- pa \frac{(1+\mu)}{E} \int_0^a J_0(mr) \cdot J_1(ma) x e^{-mz} dm$$

$$I_6: pa \int_0^a \frac{J_0(mr) \cdot J_1(ma)}{m} e^{-mz} dm = \frac{p}{2\pi} \int_0^{2\pi} \int_0^a \frac{p dp d\theta}{(x^2+r^2+p^2-2rp \cos \theta)^{1/2}}$$

$$I_7: pa \int_0^a J_0(mr) \cdot J_1(ma) x e^{-mz} dm = \frac{p}{2\pi} \int_0^{2\pi} \int_0^a \frac{z^2 p \cdot dp \cdot d\theta}{(x^2+r^2+p^2-2rp \cos \theta)^{3/2}}$$

### 2.4.7. Relations for the horizontal displacement $u$ .

$$u = pa \frac{(1+\mu)(1-2\mu)}{E} \int_0^a \frac{J_1(mr) J_1(ma)}{m} e^{-mz} dm$$

$$- pa \frac{(1+\mu)}{E} \int_0^a J_1(mr) \cdot J_1(ma) 2 e^{-mz} dm$$

$$I_8: pa \int_0^a \frac{J_1(mr) J_1(ma)}{m} e^{-mz} dm = \frac{p}{2\pi} \int_0^{2\pi} \int_0^a \frac{[(z^2 + r^2 + p^2 - 2rp \cos \theta)^{1/2} - z] p dp d\theta}{(r^2 + p^2 - 2rp \cos \theta)^{1/2} (z^2 + r^2 + p^2 - 2rp \cos \theta)^{1/2}}$$

$$I_9: pa \int_0^a J_1(mr) \cdot J_1(ma) 2 e^{-mz} dm = \frac{p}{2\pi} \int_0^{2\pi} \int_0^a \frac{2(r^2 + p^2 - 2rp \cos \theta)^{1/2} p dp d\theta}{(z^2 + r^2 + p^2 - 2rp \cos \theta)^{3/2}}$$

### 2.4.8. Resolution of the double integrals.

The integrals of alinea 2.4.7. are most easily solved in transforming the variables  $\theta$  and  $p$  by setting

$$x = p \cos \theta \quad y = p \sin \theta \quad p dp d\theta = dx dy$$

One obtains

$$I_1 = \frac{p}{2\pi} \int_{-a}^a \frac{2z(a^2 - x^2)^{1/2}}{(z^2 + x^2 + r^2 - 2xr)(z^2 + r^2 + a^2 - 2xr)^{1/2}} dx$$

that can easily be computed numerically.

$$I_2 = \frac{p}{2\pi} \int_{-a}^a \frac{2z(r^2 + x^2 - 2rx)(a^2 - x^2)^{1/2}}{(z^2 + x^2 + r^2 - 2xr)(z^2 + r^2 + a^2 - 2xr)^{1/2}} \left[ \frac{1}{(r^2 + a^2 - 2xr + z^2)} + \frac{2}{(z^2 + x^2 + r^2 - 2rx)} \right] dx$$

$$+ \frac{p}{2\pi} \int_{-a}^a \frac{2z(a^2 - x^2)^{3/2}}{(z^2 + x^2 + r^2 - 2xr)(z^2 + r^2 + a^2 - 2xr)^{3/2}} dx$$

$$- \frac{p}{2\pi} \int_{-a}^a \frac{4z(a^2 - x^2)^{1/2}}{(z^2 + x^2 + r^2 - 2xr)(z^2 + r^2 + a^2 - 2xr)^{1/2}} dx$$

$I_3$  and  $I_4$  are particular Lipschitz-Hankel integrals that we shall solve in next alinea.

$I_5$  can be deduced from  $I_2$ .

$$I_6 = \frac{p}{2\pi} \int_{-a}^a \ln \frac{(r^2 - 2xr + a^2 + z^2)^{1/2} + (a^2 - x^2)^{1/2}}{(r^2 - 2xr + a^2 + z^2)^{1/2} - (a^2 - x^2)^{1/2}} dx$$

$I_7$  can be deduced from  $I_1$ .

$I_8$  and  $I_9$  are particular Lipschitz-Hankel integrals.

#### 2.4.9. Resolution of the Lipschitz-Hankel integrals.

The solution of the Lipschitz-Hankel integrals is given by WATSON (1960).

$$\int_0^\infty e^{-at} J_\nu(bt) \cdot J_\mu(ct) t^{\nu-1} dt = \frac{(bc)^\nu \Gamma(\mu+2\nu)}{\pi a^{\mu+2\nu} \Gamma(2\nu+1)} \int_0^\pi F\left(\frac{\mu+2\nu}{2}, \frac{\mu+2\nu+1}{2}; \nu+1; -\frac{\omega^2}{a^2}\right) \sin^{2\nu} \phi d\phi$$

where

$$\omega^2 = b^2 + c^2 - 2bc \cos \phi$$

$$\begin{aligned} I_3 &= \frac{pa}{r} \int_0^\infty \frac{J_1(mr) \cdot J_1(ma)}{m} e^{-mz} dm \\ &= p \frac{a^2}{z^2} \cdot \frac{1}{2\pi} \int_0^\pi F\left(1, \frac{3}{2}; 2; -\frac{\omega^2}{z^2}\right) \sin^2 \phi d\phi \end{aligned}$$

with (WAYLAND, 1970)

$$F\left(1, \frac{3}{2}; 2; -\frac{\omega^2}{z^2}\right) = \frac{z^2}{\omega^2} \left[ 1 - \left(1 + \frac{\omega^2}{z^2}\right)^{-1/2} \right]$$

and

$$\omega^2 = a^2 + r^2 - 2ar \cos \phi$$

The resulting integral can easily be solved numerically.

$$I_4 = \frac{paz}{r} \int_0^{\infty} J_1(mr) \cdot J_1(ma) dm$$

$$= \frac{p \frac{a^2}{z^2} \cdot \frac{1}{\pi}}{\pi} \int_0^{\pi} F\left(\frac{3}{2}, 2; 2; -\frac{w^2}{z^2}\right) m^2 \phi d\phi$$

$$F\left(\frac{3}{2}, 2; 2; -\frac{w^2}{z^2}\right) = \left(1 + \frac{w^2}{z^2}\right)^{-3/2}$$

$I_8$  can be deduced from  $I_3$

$I_9$  can be deduced from  $I_4$ ;

#### 2.4.10. Expressions for computations in the axle of the load.

When stresses and displacements are computed in the axle of the load, one has

$$I_1: pa \int_0^{\infty} J_0(ma) e^{-mz} dm = \frac{\sqrt{a^2 + z^2} - z}{a \cdot \sqrt{a^2 + z^2}}$$

$$I_2: pa \int_0^{\infty} J_1(ma) m z e^{-mz} dm = \frac{az}{(a^2 + z^2)^{3/2}}$$

$$I_3: pa \int_0^{\infty} \frac{J_1(mr) \cdot J_1(ma)}{mr} e^{-mz} dm = \frac{1}{2} \cdot pa \int_0^{\infty} J_1(ma) e^{-mz} dm = \frac{I_1}{2}$$

$$I_4: pa \int_0^{\infty} \frac{J_1(mr)}{r} \cdot J_1(ma) z e^{-mz} dm = \frac{1}{2} pa \int_0^{\infty} J_1(ma) m z e^{-mz} dm = \frac{I_2}{2}$$

$$I_5: pa \int_0^{\infty} J_1(mr) J_1(ma) m z e^{-mz} dm = 0$$

$$I_6: pa \int_0^{\infty} \frac{J_1(ma)}{m} e^{-mz} dm = \frac{\sqrt{a^2 + z^2} - z}{a}$$

$$I_7: pa \int_0^{\infty} J_1(ma) z e^{-mz} dm = z \cdot I_1$$

$$I_8: pa \int_0^{\infty} \frac{J_1(mr) J_1(ma)}{m} e^{-mz} dm = 0$$

$$I_9: pa \int_0^{\infty} J_1(mr) J_1(ma) z e^{-mz} dm = 0$$

To be applicable, all the developments of paragraph 2.4. again require all the equations to be available in closeform.

### 3. The complete algebraical solution.

We shall develop here the complete algebraical analysis leading to the different computerprograms in which, because of the particular method that we have adopted, all the problems detailed above are solved.

But to make the understanding of the method easier we first develop the analysis, with the necessary comments, for a three-layer system.

This will enable us in the same time to verify the results obtained with our original method with those of existing programs.

#### 3.1. Algebraical analysis of a three-layer system (isotropic, full friction).

##### 3.1.1. Boundary conditions of the system.

We consider  $\mu_1 = \mu_2 = \mu_3 = 0.5$

$H_1$  the thickness of the first layer

$H_2$  the thickness of the second layer

We write  $A_i$  for  $A_i m^2$

$B_i$  for  $B_i m^2$

$C_i$  for  $C_i m$

$D_i$  for  $D_i m$

$$\text{and } K = \frac{E_1(1+\mu_2)}{E_2(1+\mu_1)} = \frac{E_1}{E_2}$$

$$L = \frac{E_2(1+\mu_3)}{E_3(1+\mu_2)} = \frac{E_2}{E_3}$$

Boundary conditions at the surface ( $z = 0$ ):

$$\begin{aligned} \sigma_z = p &: A_1 + B_1 = 1 \\ \tau_{rz} = 0 &: A_1 - B_1 + C_1 + D_1 = 0 \end{aligned}$$

Boundary conditions at the first interface ( $z = H_1, x = mH_1$ ):

$$\sigma_z: A_1 e^x + B_1 e^{-x} + x C_1 e^x + x D_1 e^{-x} = A_2 e^x + B_2 e^{-x} + x C_2 e^x + x D_2 e^{-x}$$

$$\tau_{rz}: A_1 e^x - B_1 e^{-x} + C_1(1+x)e^x + D_1(1-x)e^{-x} = A_2 e^x - B_2 e^{-x} + C_2(1+x)e^x + D_2(1-x)e^{-x}$$

$$w: A_1 e^x - B_1 e^{-x} + x C_1 e^x - x D_1 e^{-x} = K[A_2 e^x - B_2 e^{-x} + x C_2 e^x - x D_2 e^{-x}]$$

$$u: A_1 e^x + B_1 e^{-x} + C_1(1+x)e^x - D_1(1-x)e^{-x} = k[A_2 e^x + B_2 e^{-x} + C_2(1+x)e^x - D_2(1-x)e^{-x}]$$

Boundary conditions at the second interface ( $z = H_1 + H_2$ ,  $y = m.(H_1 + H_2)$ ):

$$\begin{aligned}\sigma_z: A_2 e^y + B_2 e^{-y} + y C_2 e^y + y D_2 e^{-y} &= B_3 e^{-y} + y D_3 e^{-y} \\ \tau_{rz}: A_2 e^y - B_2 e^{-y} + C_2 (1+y) e^y + D_2 (1-y) e^{-y} &= -B_3 e^{-y} - D_3 (1-y) e^{-y} \\ w: A_2 e^y - B_2 e^{-y} + y C_2 e^y - y D_2 e^{-y} &= L [-B_3 e^{-y} - y D_3 e^{-y}] \\ u: A_2 e^y + B_2 e^{-y} + C_2 (1+y) e^y - D_2 (1-y) e^{-y} &= L [B_3 e^{-y} - D_3 (1-y) e^{-y}]\end{aligned}$$

The boundary conditions can be written in matrixform

At the surface

$$I (A, B, C, D)_1^T = (1 \ 0)^T$$

where

$$I = \begin{pmatrix} 1 & 1 & 0 & 0 \\ 1 & -1 & 1 & 1 \end{pmatrix}$$

At the first interface

$$M_1 (A, B, C, D)_1^T = M_2 (A_2 B_2 C_2 D_2)^T$$

At the second interface

$$M_3 (A_2 B_2 C_2 D_2)^T = M_4 (B_3 D_3)^T$$

### 3.1.2. Solution of the system of 10 equations.

We start from the conditions at the second interface and write

$$(A_2 B_2 C_2 D_2)^T = M_3^{-1} \cdot M_4 (B_3 D_3)^T \quad (1)$$

Matrix  $M_3$  is very easy to invert:

$$M_3^{-1} = -\frac{1}{4} \begin{pmatrix} -2(1+y)e^{-y} & 2ye^{-y} & -2(1+y)e^{-y} & 2ye^{-y} \\ -2(1-y)e^{-y} & 2ye^y & 2(1-y)e^y & -2ye^y \\ 2e^{-y} & -2e^{-y} & 2e^{-y} & -2e^{-y} \\ -2e^y & -2e^y & 2e^y & 2e^y \end{pmatrix}$$

that we write as follows

$$M_3^{-1} = -\frac{1}{2} [e^{-y} M_{31} + e^y M_{32}]$$



with

$$M_{31} = \begin{pmatrix} -(1+\gamma) & \gamma & -(1+\gamma) & \gamma \\ 0 & 0 & 0 & 0 \\ 1 & -1 & 1 & -1 \\ 0 & 0 & 0 & 0 \end{pmatrix}$$

$$M_{32} = \begin{pmatrix} 0 & 0 & 0 & 0 \\ -(1-\gamma) & \gamma & (1-\gamma) & -\gamma \\ 0 & 0 & 0 & 0 \\ -1 & -1 & 1 & 1 \end{pmatrix}$$

and

$$M_4 = e^{-\gamma} \cdot M_{41}$$

with

$$M_{41} = \begin{pmatrix} 1 & \gamma \\ -1 & (1-\gamma) \\ -L & -L\gamma \\ L & -L(1-\gamma) \end{pmatrix}$$

so that

$$(A_2 B_2 C_2 D_2)^T = -\frac{1}{2} [e^{-2\gamma} \cdot M_{31} \cdot M_{41} + M_{32} \cdot M_{41}] (B_3 D_3)^T \quad (2)$$

We notice that the positive exponent  $e^{\gamma}$  has disappeared

We now develop those matrixproducts necessary to avoid convergency and overflowproblems. For the others we only take notice of the terms equal to zero.

So that

$$M_{31} \cdot M_{41} = \begin{pmatrix} + & + \\ 0 & 0 \\ + & + \\ 0 & 0 \end{pmatrix} = M_{3141}$$

where the sign + denotes some constant or some linear function of m.

$$M_{32} \cdot M_{41} = -(1+L) \begin{pmatrix} 0 & 0 \\ 1 & 0 \\ 0 & 0 \\ 0 & 1 \end{pmatrix} = -(1+L) \cdot U_L$$

and

$$(A_2 B_2 C_2 D_2)^T = -\frac{1}{2} [e^{-2y} M_{3141} - (1+L) \cdot U_L] (B_3 D_3)^T \quad (3)$$

The term  $(1+L) \cdot U_L$  will be part of the constant in the final denominator, constant whose value we have to know to ensure convergency at the surface.

We now consider the conditions at the first interface:

$$(A, B, C, D)^T = M_1^{-1} \cdot M_2 (A_2 B_2 C_2 D_2)^T \quad (4)$$

Matrix  $M_1^{-1}$  is identic to matrix  $M_3^{-1}$  in which  $y$  is replaced by  $x$ :

$$M_1^{-1} = -\frac{1}{2} [e^{-x} M_{11} + e^x M_{12}]$$

We write matrix  $M_2$  as follows

$$M_2 = e^x \begin{pmatrix} 1 & 0 & x & 0 \\ 1 & 0 & (1+x) & 0 \\ k & 0 & kx & 0 \\ K & 0 & K(1+x) & 0 \end{pmatrix} + e^{-x} \begin{pmatrix} 0 & 1 & 0 & x \\ 0 & -1 & 0 & (1-x) \\ 0 & -k & 0 & -1-x \\ 0 & k & 0 & -k(1-x) \end{pmatrix}$$

$$= e^x M_{21} + e^{-x} M_{22}$$

and

$$(A, B, C, D)^T = -\frac{1}{2} [(M_{11} \cdot M_{21} + M_{12} \cdot M_{22}) + e^{-2x} M_{11} \cdot M_{22} + e^{2x} M_{12} \cdot M_{21}] \cdot (A_2 B_2 C_2 D_2)^T \quad (5)$$

We again develop the products as before:

$$M_{31} \cdot M_{42} = \begin{pmatrix} 0 & + & 0 & + \\ 0 & 0 & 0 & 0 \\ 0 & + & 0 & + \\ 0 & 0 & 0 & 0 \end{pmatrix} = M_{3142}$$

$$M_{32} \cdot M_{41} = \begin{pmatrix} 0 & 0 & 0 & 0 \\ + & 0 & + & 0 \\ 0 & 0 & 0 & 0 \\ + & 0 & + & 0 \end{pmatrix} = M_{3241}$$

$$(A_2 B_2 C_2 D_2)^T = \frac{1}{16(1-t)(1-t^3)} \left[ e^{-2y} M_{3142} + e^{2y} M_{3241} + (M_{31} \cdot M_{41} + M_{32} \cdot M_{42}) \right] \\ \cdot \left[ e^{-2t} M_{5161} + e^{-2z} M_{5162} + e^{-2(t-z)} M_{5261} + M_{52} \cdot M_{62} \right] (B_4 D_4)^T$$

The products which will not converge normally are

$$e^{2y} \cdot e^{-2(t-z)} \cdot M_{3241} \cdot M_{5261}$$

because it can happen that  $y > t-z$   
and

$$e^{2y} \cdot M_{3241} \cdot M_{52} \cdot M_{62}$$

One easily verifies that

$$M_{3241} \cdot M_{5261} = 0$$

$$M_{3241} \cdot M_{52} \cdot M_{62} = 0$$

so that the concerned products disappear from the relation.

One verifies also that

$$M_{3142} \cdot M_{5161} = 0$$

$$M_{3142} \cdot M_{5162} = 0$$

$$M_4 = e^y \begin{pmatrix} 1 & 0 & -(1-2\mu_3-y) & 0 \\ 1 & 0 & (2\mu_3+y) & 0 \\ k & 0 & -k(2-4\mu_3-y) & 0 \\ k & 0 & k(1+y) & 0 \end{pmatrix} \\ + e^{-y} \begin{pmatrix} 0 & 1 & 0 & (1-2\mu_3+y) \\ 0 & -1 & 0 & (2\mu_3-y) \\ 0 & -k & 0 & -k(2-4\mu_3+y) \\ 0 & k & 0 & -k(1-y) \end{pmatrix}$$

$$(A_2 B_2 C_2 D_2)^T = -\frac{1}{4(1-\mu_1)} \left[ e^{-y} M_{31} + e^y M_{32} \right] \left[ e^y M_{41} + e^{-y} M_{42} \right] (A_3 B_3 C_3 D_3)^T \\ = -\frac{1}{4(1-\mu_1)} \left[ M_{31} \cdot M_{41} + e^{-2y} M_{31} \cdot M_{42} + e^{2y} M_{32} \cdot M_{41} + M_{32} \cdot M_{42} \right] \\ \cdot (A_3 B_3 C_3 D_3)^T$$

$$M_{31} \cdot M_{41} + M_{32} \cdot M_{42} =$$

$$\begin{pmatrix} k_1 & 0 & -k_3 & 0 \\ 0 & k_1 & 0 & k_3 \\ 0 & 0 & k_2 & 0 \\ 0 & 0 & 0 & k_2 \end{pmatrix} + y(k_1 - k_2) \begin{pmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \end{pmatrix}$$

The first matrix is a constant, the second is a linear function of  $y$ .

$$M_{52} \cdot M_{62} = \begin{pmatrix} 0 & 0 \\ L_1 & L_3 \\ 0 & 0 \\ 0 & L_2 \end{pmatrix} + 2(L_1 - L_2) \begin{pmatrix} 0 & 0 \\ 0 & 1 \\ 0 & 0 \\ 0 & 0 \end{pmatrix}$$

where

$$L_1 = -2(1-2\mu_3) - (1+L)$$

$$L_2 = -2L(1-2\mu_4) - (1+L)$$

$$L_3 = \frac{1}{2} [L_1(1-4\mu_4) - L_2(1-4\mu_3)]$$

$$(A_3 B_3 C_3 D_3)^T = -\frac{1}{4(1-\mu_3)} \left[ e^{-2t} M_{5161} + e^{-2z} M_{5162} + e^{-2(t-z)} M_{5261} + M_{52} \cdot M_{62} \right] \cdot (B_4 D_4)^T \quad (\text{Sol 1})$$

The three first terms of the expression between brackets converge normally; the last product  $M_{52} \cdot M_{62}$  contains a constant and a term, linear function of  $z$ .

We now write the conditions at the second interface in matrixform:

$$(A_2 B_2 C_2 D_2)^T = M_3^{-1} \cdot M_4 \cdot (A_3 B_3 C_3 D_3)^T$$

$$M_3^{-1} = -\frac{1}{4(1-\mu_2)} e^{-y} \begin{pmatrix} -(1+y) & -(2-4\mu_2-y) & -(2\mu_2+y) & -(1-2\mu_2-y) \\ 0 & 0 & 0 & 0 \\ 1 & -1 & 1 & -1 \\ 0 & 0 & 0 & 0 \end{pmatrix}$$

$$-\frac{1}{4(1-\mu_2)} e^y \begin{pmatrix} 0 & 0 & 0 & 0 \\ -(1-y) & (2-4\mu_2+y) & (2\mu_2-y) & -(1-2\mu_2+y) \\ 0 & 0 & 0 & 0 \end{pmatrix}$$

$$M_6 = e^{-u} \begin{pmatrix} 1 & (2-4\mu_4+t) \\ 1 & (2-4\mu_4+t) \\ L & L(2-4\mu_4+t) \\ L & L(2-4\mu_4+t) \end{pmatrix} + e^{-2} \begin{pmatrix} 1 & (1-2\mu_4+2) \\ -1 & (2\mu_4-2) \\ L & -L(2-4\mu_4+2) \\ -L & -L(1-2) \end{pmatrix}$$

$$(A_0 B_3 C_3 D_3)^T = -\frac{1}{4(1-\mu_1)} \left[ e^{-3} M_{51} + e^{-2} M_{52} \right] \cdot \left[ e^{-u} M_{61} + e^{-2} M_{62} \right]$$

$$M_{51} \cdot M_{61} = \begin{pmatrix} + & + \\ 0 & 0 \\ + & + \\ 0 & 0 \end{pmatrix} = M_{5161}$$

$$M_{51} \cdot M_{62} = \begin{pmatrix} + & + \\ 0 & 0 \\ + & + \\ 0 & 0 \end{pmatrix} = M_{5162}$$

$$M_{52} \cdot M_{61} = \begin{pmatrix} 0 & 0 \\ + & + \\ 0 & 0 \\ + & + \end{pmatrix} = M_{5261}$$

$$w: A_3 e^z - B_3 e^{-z} - C_3 (2-4\mu_3-2) e^z - D_3 (2-4\mu_3+2) e^{-z}$$

$$L_w [A_4 e^z - B_4 e^{-z} - C_4 (2-4\mu_4-2) e^z - D_4 (2-4\mu_4+2) e^{-z}]$$

$$u: A_3 e^z + B_3 e^{-z} + C_3 (1+2) e^z - D_3 (1-2) e^{-z}$$

$$L_u [A_4 e^z + B_4 e^{-z} + C_4 (1+2) e^z - D_4 (1-2) e^{-z}]$$

Boundary conditions at the bottom ( $z = H_1 + H_2 + H_3 + H_4$ ):

$$w: A_4 e^t - B_4 e^{-t} - C_4 (2-4\mu_4-t) e^t - D_4 (2-4\mu_4+t) e^{-t} = 0$$

Isotropic:  $C_4 = 0$ .

### 3.2.2. Solution of the system of 16 equations:

In the equations of the conditions at the third interface,  $A_4$  is replaced by its value taken from the fixed bottom condition:

$$A_4 e^t = B_4 e^{-t} + D_4 (2-4\mu_4+t) e^{-t}$$

We write the conditions at the third interface in matrix form

$$(A_3 B_3 C_3 D_3)^T = M_3^{-1} \cdot M_6 (B_4 D_4)^T$$

For simplicity here, we take  $F_w = F_u = F$ ,  $k_w = k_u = k$ ,  $L_w = L_u = L$

so that  $k_1 = k_2 = k_3 = 1$

$$M_3^{-1} = -\frac{1}{4(1-\mu_3)} e^{-2} \begin{pmatrix} -(1+2) & -(2-4\mu_3-2) & -(2\mu_3+2) & -(1-2\mu_3-2) \\ 0 & 0 & 0 & 0 \\ 1 & -1 & 1 & -1 \\ 0 & 0 & 0 & 0 \end{pmatrix}$$

$$-\frac{1}{4(1-\mu_3)} e^2 \begin{pmatrix} 0 & 0 & 0 & 0 \\ -(1-2) & (2-4\mu_3+2) & (2\mu_3-2) & -(1-2\mu_3+2) \\ 0 & 0 & 0 & 0 \\ -1 & -1 & 1 & 1 \end{pmatrix}$$

Boundary conditions at the first interface ( $z = H_1$ ):

$$\begin{aligned}
 \sigma_z: & A_1 e^x + B_1 e^{-x} - C_1 (1-2\mu_1 - x) e^x + D_1 (1-2\mu_1 + x) e^{-x} = \\
 & A_2 e^x + B_2 e^{-x} - C_2 (1-2\mu_2 - x) e^x + D_2 (1-2\mu_2 + x) e^{-x} \\
 \tau_{rz}: & A_1 e^x - B_1 e^{-x} + C_1 (2\mu_1 + x) e^x + D_1 (2\mu_1 - x) e^{-x} = \\
 & A_2 e^x - B_2 e^{-x} + C_2 (2\mu_2 + x) e^x + D_2 (2\mu_2 - x) e^{-x} \\
 w: & A_1 e^x - B_1 e^{-x} - C_1 (2-4\mu_1 + x) e^x - D_1 (2-4\mu_1 + x) e^{-x} = \\
 & F_w [A_2 e^x - B_2 e^{-x} - C_2 (2-4\mu_2 + x) e^x - D_2 (2-4\mu_2 + x) e^{-x}] \\
 u: & A_1 e^x + B_1 e^{-x} + C_1 (1+x) e^x - D_1 (1-x) e^{-x} = \\
 & F_u [A_2 e^x + B_2 e^{-x} + C_2 (1+x) e^x - D_2 (1-x) e^{-x}]
 \end{aligned}$$

Boundary conditions at the second interface ( $z = H_1 + H_2$ ):

$$\begin{aligned}
 \sigma_z: & A_2 e^y + B_2 e^{-y} - C_2 (1-2\mu_2 - y) e^y + D_2 (1-2\mu_2 + y) e^{-y} = \\
 & A_3 e^y + B_3 e^{-y} - C_3 (1-2\mu_3 - y) e^y + D_3 (1-2\mu_3 + y) e^{-y} \\
 \tau_{rz}: & A_2 e^y - B_2 e^{-y} + C_2 (2\mu_2 + y) e^y + D_2 (2\mu_2 - y) e^{-y} = \\
 & A_3 e^y - B_3 e^{-y} + C_3 (2\mu_3 + y) e^y + D_3 (2\mu_3 - y) e^{-y} \\
 w: & A_2 e^y - B_2 e^{-y} - C_2 (2-4\mu_2 - y) e^y - D_2 (2-4\mu_2 + y) e^{-y} = \\
 & K_w [A_3 e^y - B_3 e^{-y} - C_3 (2-4\mu_3 - y) e^y - D_3 (2-4\mu_3 + y) e^{-y}] \\
 u: & A_2 e^y + B_2 e^{-y} + C_2 (1+y) e^y - D_2 (1-y) e^{-y} = \\
 & K_u [A_3 e^y + B_3 e^{-y} + C_3 (1+y) e^y - D_3 (1-y) e^{-y}]
 \end{aligned}$$

Boundary conditions at the third interface ( $z = H_1 + H_2 + H_3$ ):

$$\begin{aligned}
 \sigma_z: & A_3 e^z + B_3 e^{-z} - C_3 (1-2\mu_3 - z) e^z + D_3 (1-2\mu_3 + z) e^{-z} = \\
 & A_4 e^z + B_4 e^{-z} - C_4 (1-2\mu_4 - z) e^z + D_4 (1-2\mu_4 + z) e^{-z} \\
 \tau_{rz}: & A_3 e^z - B_3 e^{-z} + C_3 (2\mu_3 + z) e^z + D_3 (2\mu_3 - z) e^{-z} = \\
 & A_4 e^z - B_4 e^{-z} + C_4 (2\mu_4 + z) e^z + D_4 (2\mu_4 - z) e^{-z}
 \end{aligned}$$



### 3.2. Algebraical analysis of an isotropic four-layer system with fixed bottom.

#### 3.2.1. Boundary conditions.

In this analysis, we choose as fixed bottom condition the one described in § 1.2.2. ( $w = 0$ ).

We write

$$A_i = A_i m^2 \quad B_i = B_i m^2 \quad C_i = C_i m \quad = D_i m$$

$$F_w = \frac{E_1(1+\mu_2)}{E_2(1+\mu_1)} \quad F_u = k_1 \frac{E_1(1+\mu_2)}{E_2(1+\mu_1)}$$

$$K_w = \frac{E_2(1+\mu_3)}{E_3(1+\mu_2)} \quad K_u = k_2 \frac{E_2(1+\mu_3)}{E_3(1+\mu_2)}$$

$$L_w = \frac{E_3(1+\mu_4)}{E_4(1+\mu_3)} \quad L_u = k_3 \frac{E_3(1+\mu_4)}{E_4(1+\mu_3)}$$

$$x = m H_1$$

$$y = m (H_1 + H_2)$$

$$z = m (H_1 + H_2 + H_3)$$

$$r = m (H_1 + H_2 + H_3 + H_4)$$

$$u = -2t + z$$

where  $H_1$ ,  $H_2$ ,  $H_3$  and  $H_4$  are the thicknesses of the four layers.

Boundary conditions at the surface ( $z = 0$ ):

$$\sigma_z = p \quad A_1 + B_1 - C_1(1-2\mu_1) + D_1(1-2\mu_1) = 1$$

$$\tau_{rz} = 0 \quad A_1 - B_1 + C_1 \cdot 2\mu_1 + D_1 \cdot 2\mu_1 = 0$$

```

E1=100000      U1=0.50      H1=10.00      A= 10.00      P= 1.00
E2= 10000      U2=0.50      H2=20.00      R=  0.00
E3=  1000      U3=0.50

*****
*              * SURFACE * BASE   * SURFACE * BASE   * SURFACE *
*              *          * COUCHE 1 * COUCHE 2 * COUCHE 2 * COUCHE 3 *
*****
*CONTRAINTES VERTICALES *      * +0.247002* +0.247002* +0.029472* +0.029472*
*****
*CONTRAINTES RADIALES  *      * -1.864850* +0.035817* -0.204340* +0.006091*
*****
*CONTRAINTES CIRCONF.  *      * -1.864850*          * -0.204340* +0.006091*
*****
*FLECHES              *+2.183E-03*          *          *          *
*****

```

Table 1. Stresses in a three-layer system (BASIC Program).

## RESULTATS DANS L'AXE DE LA CHARGE.

```

E1=100000.      U1=0.50      H1=10.00      A= 10.00      P= 1.00
E2= 10000.      U2=0.50      H2=20.00      R=  .00
E3=  1000.      U3=0.50

              SURFACE      BASE      SURFACE      BASE      SURFACE
              COUCHE1      COUCHE2      COUCHE2      COUCHE3

CONTRAINTES VERTICALES  *      .247002      .247002      .029472      .029472
CONTRAINTES RADIALES    *      -1.864850      .035817      -.204340      .006091
CONTRAINTES CIRCONF.    *      -1.864850      .035817      -.204340      .006091
FLECHES                  .0022      *          *          *

```

Table 2. Stresses in a three-layer system (FORTRAN 77 Program).

#### 3.1.4. Comparison with existing programs.

First we have computed the stresses in a three-layer isotropic system with a program written in BASIC and for which the equations had been developed in complete closeform (VAN CAUWELAERT, 1983).

The accuracy of this program had been checked earlier with the results published by JONES (1962 ).

Then we have written a new program in FORTRAN 77 based on the developments presented in paragraphs 3.1.1. and 3.1.2. and calculated with this program the stresses in the same three-layer system.

The results obtained by the program in BASIC are given in table 1, those obtained by the program in FORTRAN 77 are given in table 2: we notice that the agreement between both is perfect.

The complete listing of the FORTRAN 77 program is given in appendix:

The matrices are dimensionned and upbuilt in the beginning of the program. The variables  $x (=mH_1)$  and  $y (=mH_1 + mH_2)$  are introduced in the matrices by instruction 214. The products between matrices are carried out in the same instruction 214 by calling subroutines 800, 810, 830 and 860.

### 3.1.3. Relation for the vertical deflection at the surface.

We notice that the values of the parameters given by (10) and (11) all converge normally, except for parameters  $B_1$  and  $D_1$  which contain each a constant in the numerator:

$$\begin{aligned} \frac{1}{4} (1+k)(1+L) B_3 &= \frac{1}{4} (1+k)(1+L) \frac{4(1+k)(1+L) + 4d_{22}}{\nabla} \\ &= \frac{[(1+k)(1+L)]^2}{\nabla} + \frac{(1+k)(1+L) \cdot d_{22}}{\nabla} \end{aligned}$$

The influence of this constant can be eliminated as indicated in § 2.3.

Here it is nevertheless easier to eliminate  $B_1$ .

From the boundary conditions at the surface, we have

$$B_1 = 1 - A_1$$

The relation for the deflection at the surface is given by

$$\begin{aligned} w &= -pa \int_0^\infty \frac{J_1(ma)}{m} \cdot \frac{1+\mu_1}{E_1} [A_1 - B_1] dm \\ &= -pa \cdot \frac{1+\mu_1}{E_1} \int_0^\infty \frac{J_1(ma)}{m} (1 - 2A_1) dm \\ &= -pa \frac{1+\mu_1}{E_1} \int_0^\infty \frac{J_1(ma)}{m} dm + 2pa \frac{1+\mu_1}{E_1} \int_0^\infty \frac{J_1(ma)}{m} A_1 dm \quad (12) \\ &= -pa \frac{1+\mu_1}{E_1} + 2pa \frac{1+\mu_1}{E_1} \int_0^\infty \frac{J_1(ma)}{m} A_1 dm \end{aligned}$$

The first integral of (12) is solved analytically, the second one converges fast and safely during the numerical integration procedure.

The parameters  $A_1$ ,  $B_1$ ,  $C_1$  and  $D_1$  are then obtained from (6)

$$\begin{aligned}
 A_1 &= -\frac{1}{4} \left\{ (1+k)e^{-2y} [M_{3141}(1,1) B_3 + M_{3141}(1,2) D_3] \right. \\
 &\quad \left. + (1+L)e^{-2x} [M_{1122L}(1,1) B_3 + M_{1122L}(1,2) D_3] \right\} \\
 B_1 &= \frac{1}{4} \left\{ (1+k)(1+L) B_3 + e^{-2(y-x)} [M_{1241}(2,1) B_3 + M_{1241}(2,2) D_3] \right\} \\
 C_1 &= -\frac{1}{4} \left\{ (1+k)e^{-2y} [M_{3141}(3,1) B_3 + M_{3141}(3,2) D_3] \right. \\
 &\quad \left. + (1+L)e^{-2x} [M_{1122L}(3,1) B_3 + M_{1122L}(3,2) D_3] \right\} \\
 D_1 &= \frac{1}{4} \left\{ (1+k)(1+L) D_3 + e^{-2(y-x)} [M_{1241}(4,1) B_3 + M_{1241}(4,2) D_3] \right\}
 \end{aligned} \tag{10}$$

Parameters  $A_2$ ,  $B_2$ ,  $C_2$  and  $D_2$  are obtained from (3)

$$\begin{aligned}
 A_2 &= -\frac{1}{2} e^{-2y} [M_{3141}(1,1) B_3 + M_{3141}(1,2) D_3] \\
 B_2 &= \frac{1}{2} (1+L) B_3 \\
 C_2 &= -\frac{1}{2} e^{-2y} [M_{3141}(3,1) B_3 + M_{3141}(3,2) D_3] \\
 D_2 &= \frac{1}{2} (1+L) D_3
 \end{aligned} \tag{11}$$

$$I. M_{3141} = \begin{pmatrix} a_{11} & a_{12} \\ a_{21} & a_{22} \end{pmatrix}$$

$$I. M_{1241} = \begin{pmatrix} b_{11} & b_{12} \\ b_{21} & b_{22} \end{pmatrix}$$

$$I. M_{1122L} = \begin{pmatrix} c_{11} & c_{12} \\ c_{21} & c_{22} \end{pmatrix}$$

The terms  $a_{ij}$ ,  $b_{ij}$  and  $c_{ij}$  are the results of very simple computerprocedures.

We can now write (8) in extended form:

$$\begin{aligned} & \left[ (1+k)(1+L) - (1+k) \cdot a_{11} \cdot e^{-2y} + b_{11} \cdot e^{-2(y-x)} - (1+L) \cdot c_{11} \cdot e^{-2x} \right] B_3 \\ & + \left[ -(1+k) \cdot a_{12} \cdot e^{-2y} + b_{12} \cdot e^{-2(y-x)} - (1+L) \cdot c_{12} \cdot e^{-2x} \right] D_3 = 4 \end{aligned}$$

$$\begin{aligned} & \left[ -(1+k)(1+L) - (1+k) \cdot a_{21} \cdot e^{-2y} + b_{21} \cdot e^{-2(y-x)} - (1+L) \cdot c_{21} \cdot e^{-2x} \right] B_3 \\ & + \left[ (1+k)(1+L) - (1+k) \cdot a_{22} \cdot e^{-2y} + b_{22} \cdot e^{-2(y-x)} - (1+L) \cdot c_{22} \cdot e^{-2x} \right] D_3 = 0 \end{aligned}$$

We write this

$$\begin{aligned} & \left[ (1+k)(1+L) + d_{11} \right] B_3 + d_{12} \cdot D_3 = 4 \\ & \left[ -(1+k)(1+L) + d_{21} \right] B_3 + \left[ (1+k)(1+L) + d_{22} \right] D_3 = 0 \end{aligned}$$

where  $(1+k) \cdot (1+L)$  is, as a matter of fact, a constant and all  $d_{11}$ ,  $d_{12}$ ,  $d_{21}$  and  $d_{22}$  converge normally.

Finally one obtains

$$B_3 = \frac{4(1+k)(1+L) + 4d_{22}}{\nabla} \quad D_3 = \frac{4(1+k)(1+L) - 4d_{11}}{\nabla}$$

where

$$\nabla = \left[ (1+k)(1+L)^2 + (1+k)(1+L)(d_{11} + d_{12} + d_{22}) + d_{11} \cdot d_{22} - d_{12} \cdot d_{21} \right]$$

$$M_{1122} \cdot M_{3141} = 0$$

$$M_{1221} \cdot M_{3141} = \begin{pmatrix} 0 & 0 \\ + & + \\ 0 & 0 \\ + & + \end{pmatrix} = M_{1241}$$

$$M_{1122} \cdot U_L = \begin{pmatrix} + & + \\ 0 & 0 \\ + & + \\ 0 & 0 \end{pmatrix} = M_{1122} L$$

$$M_{1221} \cdot U_L = 0$$

so that

$$(A, B, C, D)^T = \frac{1}{4} \left[ (1+k)(1+L) \cdot U_L - (1+k)e^{-2y} M_{3141} + e^{-2(y-x)} M_{1241} - (1+L)e^{-2x} M_{1122} L \right] (B_3 \ D_3)^T \quad (6)$$

The positive exponent  $e^{2x}$  has again disappeared together with the highest negative exponent  $e^{-2(y+x)}$ .

Finally we consider the conditions at the surface

$$I(A, B, C, D)^T = \begin{pmatrix} 1 & 0 \end{pmatrix}^T \quad (7)$$

We replace in (7),  $(A_1 \ B_1 \ C_1 \ D_1)^T$  by its value from (6)

$$\frac{1}{4} I \left[ (1+k)(1+L) \cdot U_L - (1+k)e^{-2y} \cdot M_{3141} + e^{-2(y-x)} \cdot M_{1241} - (1+L)e^{-2x} \cdot M_{1122} L \right] (B_3 \ D_3)^T = \begin{pmatrix} 1 & 0 \end{pmatrix}^T \quad (8)$$

$$(1+k)(1+L) I \cdot U_L = (1+k)(1+L) \begin{pmatrix} 1 & 0 \\ -1 & 1 \end{pmatrix}$$

$$M_{11} \cdot M_{21} + M_{12} \cdot M_{22} = -(1+k) \begin{pmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{pmatrix} = -(1+k) U_k$$

$$M_{11} \cdot M_{22} = \begin{pmatrix} 0 & + & 0 & + \\ 0 & 0 & 0 & 0 \\ 0 & + & 0 & + \\ 0 & 0 & 0 & 0 \end{pmatrix} = M_{1122}$$

$$M_{12} \cdot M_{21} = \begin{pmatrix} 0 & 0 & 0 & 0 \\ + & 0 & + & 0 \\ 0 & 0 & 0 & 0 \\ + & 0 & + & 0 \end{pmatrix} = M_{1221}$$

We replace in (5),  $(A_2 \ B_2 \ C_2 \ D_2)^T$  by its value from (3)

$$\begin{aligned} (A, B, C, D)^T &= \frac{1}{4} \left[ -(1+k) \cdot U_k + e^{-2x} \cdot M_{1122} + e^{2x} M_{1221} \right] \cdot \\ &\quad \left[ e^{-2y} M_{3141} - (1+L) U_L \right] (B_3 \ D_3)^T \\ &= \frac{1}{4} \left[ (1+k)(1+L) U_k \cdot U_L - (1+k) e^{-2y} U_k \cdot M_{3141} \right. \\ &\quad \left. + e^{-2(y+x)} \cdot M_{1122} \cdot M_{3141} + e^{-2(y-x)} \cdot M_{1221} \cdot M_{3141} \right. \\ &\quad \left. - (1+L) e^{-2x} M_{1122} \cdot U_L - (1+L) e^{2x} M_{1221} \cdot U_L \right] (B_3 \ D_3)^T \end{aligned}$$

$$U_k \cdot U_L = \begin{pmatrix} 0 & 0 \\ 1 & 0 \\ 0 & 0 \\ 0 & 1 \end{pmatrix} = U_L$$

$$U_k \cdot M_{3141} = \begin{pmatrix} + & + \\ 0 & 0 \\ + & + \\ 0 & 0 \end{pmatrix} = M_{3141}$$



The product we are interested in for later computation is

$$\begin{aligned}
 & (M_{31} \cdot M_{41} + M_{32} \cdot M_{42}) \cdot (M_{52} \cdot M_{62}) = \\
 & \begin{pmatrix} 0 & 0 \\ \kappa_1 L_1 & \kappa_1 L_3 + \kappa_3 L_2 \\ 0 & 0 \\ 0 & \kappa_2 L_2 \end{pmatrix} + \gamma (\kappa_1 - \kappa_2) \begin{pmatrix} 0 & 0 \\ 0 & L_2 \\ 0 & 0 \\ 0 & 0 \end{pmatrix} \\
 & + z(L_1 - L_2) \begin{pmatrix} 0 & 0 \\ 0 & \kappa_1 \\ 0 & 0 \\ 0 & 0 \end{pmatrix} + \gamma z (\kappa_1 - \kappa_2) (L_1 - L_2) \begin{pmatrix} 0 & 0 \\ 0 & 0 \\ 0 & 0 \\ 0 & 0 \end{pmatrix} = N_{41}
 \end{aligned}$$

This product contains a constant, a linear function of  $y$  and a linear function of  $z$ . The term, function of  $y \cdot z$ , a nonlinear function, disappears.

The other products are

$$M_{3142} \cdot M_{5261} = \begin{pmatrix} + & + \\ 0 & 0 \\ + & + \\ 0 & 0 \end{pmatrix} = N_{21}$$

$$M_{3142} \cdot M_{52} \cdot M_{62} = \begin{pmatrix} + & + \\ 0 & 0 \\ + & + \\ 0 & 0 \end{pmatrix} = N_{22}$$

$$M_{3241} \cdot M_{5161} = \begin{pmatrix} 0 & 0 \\ + & + \\ 0 & 0 \\ + & + \end{pmatrix} = N_{12}$$

$$M_{2241} \cdot M_{5162} = \begin{pmatrix} 0 & 0 \\ + & + \\ 0 & 0 \\ + & + \end{pmatrix} = N_{13}$$

$$(M_{31} \cdot M_{41} + M_{32} \cdot M_{42}) \cdot M_{5161} = \begin{pmatrix} + & + \\ 0 & 0 \\ + & + \\ 0 & 0 \end{pmatrix} = N_{23}$$

$$(M_{31} \cdot M_{41} + M_{32} \cdot M_{42}) \cdot M_{5162} = \begin{pmatrix} + & + \\ 0 & 0 \\ + & + \\ 0 & 0 \end{pmatrix} = N_{24}$$

$$(M_{31} \cdot M_{41} + M_{32} \cdot M_{42}) \cdot M_{5261} = \begin{pmatrix} 0 & 0 \\ + & + \\ 0 & 0 \\ + & + \end{pmatrix} = N_{14}$$

with

$$N_{1i} = \begin{pmatrix} 0 & 0 \\ + & + \\ 0 & 0 \\ + & + \end{pmatrix}$$

$$N_{2i} = \begin{pmatrix} + & + \\ 0 & 0 \\ + & + \\ 0 & 0 \end{pmatrix}$$

The final relation becomes

$$(A_2 B_2 C_2 D_2)^T = \frac{1}{16(1-\mu_2)(1-\mu_3)}$$

$$\begin{aligned} & \left[ e^{-2y} \cdot e^{-2(r-z)} \cdot N_{21} + e^{-2y} \cdot N_{22} + e^{-2(r-y)} \cdot N_{12} \right. \\ & \quad + e^{-2(z-y)} \cdot N_{13} + e^{-2r} \cdot N_{23} + e^{-2z} \cdot N_{24} \\ & \quad \left. + e^{-2(r-z)} \cdot N_{14} + N_{11} \right] (B_4 D_4)^T \quad (\text{sol. 2}) \end{aligned}$$

Next we write the conditions at the first interface in matrix form

$$(A, B, C, D)_1^T = M_1^{-1} \cdot M_2 (A_2 B_2 C_2 D_2)^T$$

$$M_1^{-1} = -\frac{1}{4(1-\mu_1)} e^{-x} \begin{pmatrix} -(1+x) & -(2-4\mu_1-x) & -(2\mu_1+x) & -(1-2\mu_1-x) \\ 0 & 0 & 0 & 0 \\ 1 & -1 & 1 & -1 \\ 0 & 0 & 0 & 0 \end{pmatrix}$$

$$-\frac{1}{4(1-\mu_1)} e^x \begin{pmatrix} 0 & 0 & 0 & 0 \\ -(1-x) & (2-4\mu_1+x) & (2\mu_1-x) & -(1-2\mu_1+x) \\ 0 & 0 & 0 & 0 \\ -1 & -1 & 1 & 1 \end{pmatrix}$$

$$M_2 = e^x \begin{pmatrix} 1 & 0 & -(1-2\mu_2-x) & 0 \\ 1 & 0 & (2\mu_2+x) & 0 \\ F & 0 & -F(2-4\mu_2-x) & 0 \\ F & 0 & F(1+x) & 0 \end{pmatrix}$$

$$+ e^{-x} \begin{pmatrix} 0 & 1 & 0 & (1-2\mu_1+x) \\ 0 & -1 & 0 & (2\mu_1-x) \\ 0 & -F & 0 & -F(2-4\mu_1+x) \\ 0 & F & 0 & -F(1-x) \end{pmatrix}$$

$$\begin{aligned} (A, B, C, D)_1^T &= -\frac{1}{4(1-\mu_1)} \left[ e^{-x} M_{11} + e^x M_{12} \right] \cdot \left[ e^x M_{21} + e^{-x} M_{22} \right] (A_2 B_2 C_2 D_2)^T \\ &= -\frac{1}{4(1-\mu_1)} \left[ M_{11} \cdot M_{21} + e^{-2x} M_{11} \cdot M_{22} + e^{2x} M_{12} \cdot M_{21} + M_{12} M_{22} \right] (A_2 B_2 C_2 D_2)^T \end{aligned}$$

$$M_{11} \cdot M_{21} + M_{12} \cdot M_{22} =$$

$$\begin{pmatrix} F_1 & 0 & -F_3 & 0 \\ 0 & F_1 & 0 & F_3 \\ 0 & 0 & F_2 & 0 \\ 0 & 0 & 0 & F_2 \end{pmatrix} + x(F_1 - F_2) \begin{pmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \end{pmatrix}$$

The first matrix is a constant, the second is a linear function of  $x$ .

$$M_{11} \cdot M_{22} = \begin{pmatrix} 0 & + & 0 & + \\ 0 & 0 & 0 & 0 \\ 0 & + & 0 & + \\ 0 & 0 & 0 & 0 \end{pmatrix} = M_{1122}$$

$$M_{12} \cdot M_{21} = \begin{pmatrix} 0 & 0 & 0 & 0 \\ + & 0 & + & 0 \\ 0 & 0 & 0 & 0 \\ + & 0 & + & 0 \end{pmatrix} = M_{1221}$$

$$(A, B, C, D)^T = - \frac{1}{b_4(1-\mu_1)(1-\mu_2)(1-\mu_3)} \cdot [M_{11} \cdot M_{21} + M_{12} \cdot M_{22} + M_{1121} e^{-2x} + M_{1221} e^{2x}]$$

$$\begin{aligned} & \left[ e^{-2y} \cdot e^{-2(r-z)} \cdot N_{21} + e^{-2y} \cdot N_{22} + e^{-2(r-y)} N_{12} \right. \\ & + e^{-2(z-y)} N_{13} + e^{-2r} N_{23} + e^{-2z} N_{24} \\ & \left. + e^{-2(r-z)} N_{14} + N_{11} \right] (B_4 \ D_4)^T \end{aligned}$$

The products which will not converge normally are

$$e^{2x} e^{-2(r-y)} M_{1221} \cdot M_{3241} \cdot M_{5161}$$

$$e^{2x} e^{-2(z-y)} M_{1221} \cdot M_{3241} \cdot M_{5162}$$

$$e^{2x} e^{-2(r-z)} M_{1221} (M_{31} \cdot M_{41} + M_{32} \cdot M_{42}) \cdot M_{5261}$$

because that it can happen that  $x > r-y$   
 or  $x > z-y$   
 or  $x > r-z$

and

$$e^{2x} M_{1221} (M_{31} \cdot M_{41} + M_{32} \cdot M_{42}) \cdot (M_{52} \cdot M_{62})$$

One easily verifies that

$$M_{1221} \cdot M_{3241} = 0$$

$$M_{1221} \cdot (M_{31} \cdot M_{41} + M_{32} \cdot M_{42}) \cdot M_{5261} = 0$$

$$M_{1221} \cdot (M_{31} \cdot M_{41} + M_{32} \cdot M_{42}) \cdot M_{52} \cdot M_{62} = 0$$

so that again the concerned products disappear from the relation.

The product, we are interested in for later computation, is

$$\begin{aligned} & [M_{11} \cdot M_{21} + M_{12} \cdot M_{22}] [M_{31} \cdot M_{41} + M_{32} \cdot M_{42}] [M_{52} \cdot M_{62}] = \\ & \begin{pmatrix} 0 & 0 \\ F_1 k_1 L_1 & F_1 k_1 L_3 + F_1 k_3 L_2 + F_3 k_2 L_2 \\ 0 & 0 \\ 0 & F_2 k_2 L_2 \end{pmatrix} + y(k_1 - k_2) \begin{pmatrix} 0 & 0 \\ 0 & F_1 L_2 \\ 0 & 0 \\ 0 & 0 \end{pmatrix} \\ & + x(F_1 - F_2) \begin{pmatrix} 0 & 0 \\ 0 & k_2 L_2 \\ 0 & 0 \\ 0 & 0 \end{pmatrix} + z(L_1 - L_2) \begin{pmatrix} 0 & 0 \\ 0 & F_1 k_1 \\ 0 & 0 \\ 0 & 0 \end{pmatrix} = T_{11} \end{aligned}$$

$$\text{One has } F_1 k_1 L_3 + F_1 k_3 L_2 + F_3 k_2 L_2 = \frac{1}{2} [k_1 L_1 F_1 (1-4\mu_1) - k_2 L_2 F_2 (1-4\mu_1)]$$

One verifies that the products

$$M_{1122} \cdot N_i = 0$$

$$M_{1221} \cdot N_i = 0$$

The other products are

$$(M_{11} \cdot M_{21} + M_{12} \cdot M_{22}) \cdot N_{1i} = \begin{pmatrix} 0 & 0 \\ + & + \\ 0 & 0 \\ + & + \end{pmatrix} = T_{1i}$$

$$(M_{11} \cdot M_{21} + M_{12} \cdot M_{22}) \cdot N_{2i} = \begin{pmatrix} + & + \\ 0 & 0 \\ + & + \\ 0 & 0 \end{pmatrix} = T_{2i}$$

$$M_{1122} \cdot N_{1i} = \begin{pmatrix} + & + \\ 0 & 0 \\ + & + \\ 0 & 0 \end{pmatrix} = T_{3i} \quad M_{1221} \cdot N_{2i} = \begin{pmatrix} 0 & 0 \\ + & + \\ 0 & 0 \\ + & + \end{pmatrix} = T_{4i}$$

and finally

$$\begin{aligned} (A, B, C, D)^T = & - \frac{1}{64(1-\mu_1)(1-\mu_2)(1-\mu_3)} \\ & \left[ e^{-2y} \cdot e^{-2(r-z)} \cdot T_{21} + e^{-2y} \cdot T_{22} + e^{-2(r-y)} \cdot T_{12} + e^{-2(z-y)} \cdot T_{13} \right. \\ & + e^{-2t} \cdot T_{23} + e^{-2z} \cdot T_{24} + e^{-2(r-z)} \cdot T_{14} + e^{-2x} \cdot e^{-2(r-y)} \cdot T_{32} \\ & + e^{-2x} \cdot e^{-2(z-y)} \cdot T_{33} + e^{-2x} \cdot e^{-2(r-z)} \cdot T_{34} + e^{-2x} \cdot T_{31} \\ & + e^{-2(y-x)} \cdot e^{-2(r-z)} \cdot T_{41} + e^{-2(y-x)} \cdot T_{42} + e^{-2(r-x)} \cdot T_{43} \\ & \left. + e^{-2(z-x)} \cdot T_{44} + T_{11} \right] (D_4 \ D_4)^T \quad (A.3) \end{aligned}$$

Finally we consider the boundary conditions at the surface:

$$\begin{pmatrix} 1 & 1 & -(1-2\mu_1) & (1-2\mu_1) \\ 1 & -1 & 2\mu_1 & 2\mu_1 \end{pmatrix} (A, B, C, D)^T = (1 \ 0)^T$$

that we can transform into

$$-\frac{1}{64(1-\mu_1)(1-\mu_2)(1-\mu_3)} \begin{pmatrix} 1 & 1 & -(1-2\mu_1) & (1-2\mu_1) \\ 1 & -1 & 2\mu_1 & 2\mu_1 \end{pmatrix} \left[ \begin{pmatrix} c_{11} & c_{12} \\ c_{21} & c_{22} \\ c_{31} & c_{32} \\ c_{41} & c_{42} \end{pmatrix} + \begin{pmatrix} 0 & 0 \\ d_{21} & d_{22} \\ 0 & 0 \\ 0 & d_{42} \end{pmatrix} \right] (B_4 \ D_4)^T = (1 \ 0)^T$$

Extending matrix  $(d_{ij})$ , equal to  $T_{11}$ , we obtain

$$-\frac{1}{64(1-\mu_1)(1-\mu_2)(1-\mu_3)} \left\{ \begin{pmatrix} 1 & 1 & -(1-2\mu_1) & (1-2\mu_1) \\ 1 & -1 & 2\mu_1 & 2\mu_1 \end{pmatrix} \begin{pmatrix} c_{11} & c_{12} \\ c_{21} & c_{22} \\ c_{31} & c_{32} \\ c_{41} & c_{42} \end{pmatrix} + \begin{pmatrix} F_1 k_1 L_1 & \frac{1}{2} [F_1 k_1 L_1 (1-4\mu_4) + F_2 k_2 L_2] \\ -F_1 k_1 L_1 & -\frac{1}{2} [F_1 k_1 L_1 (1-4\mu_4) - F_2 k_2 L_2] \end{pmatrix} + \left[ x(F_1 - F_2) k_2 L_2 + y(k_1 - k_2) F_1 L_2 + z(L_1 - L_2) F_1 k_1 \right] \begin{pmatrix} 0 & 1 \\ 0 & -1 \end{pmatrix} \right\} (B_4 \ D_4)^T = (1 \ 0)^T$$

This last relation can be written in extended form

$$\left[ c_{11} + c_{21} - (1-2\mu_1) c_{31} + (1-2\mu_1) c_{41} + F_1 k_1 L_1 \right] B_4 \\ + \left\{ c_{12} + c_{22} - (1-2\mu_1) c_{32} + (1-2\mu_1) c_{42} + \frac{1}{2} [F_1 k_1 L_1 (1-4\mu_4) + F_2 k_2 L_2] \right. \\ \left. + x(F_1 - F_2) k_2 L_2 + y(k_1 - k_2) F_1 L_2 + z(L_1 - L_2) F_1 k_1 \right\} = - \frac{(1-\mu_1)(1-\mu_2)(1-\mu_3)}{64}$$

$$\left[ c_{11} - c_{21} + 2\mu_1 c_{31} + 2\mu_1 c_{41} - F_1 k_1 L_1 \right] B_4 \\ + \left\{ c_{12} - c_{22} + 2\mu_1 c_{32} + 2\mu_1 c_{42} - \frac{1}{2} [F_1 k_1 L_1 (1-4\mu_4) - F_2 k_2 L_2] \right. \\ \left. - x(F_1 - F_2) k_2 L_2 - y(k_1 - k_2) F_1 L_2 - z(L_1 - L_2) F_1 k_1 \right\} = 0$$

We can write this system as follows

$$(n_{11} + F_1 k_1 L_1) B_4 + \left\{ n_{12} + \frac{1}{2} [F_1 k_1 L_1 (1-4\mu_4) + F_2 k_2 L_2] + f(xyz) \right\} D_4 = \\ = - \frac{(1-\mu_1)(1-\mu_2)(1-\mu_3)}{64}$$

$$(n_{21} - F_1 k_1 L_1) B_4 + \left\{ n_{22} - \frac{1}{2} [F_1 k_1 L_1 (1-4\mu_4) - F_2 k_2 L_2] - f(xyz) \right\} D_4 = 0$$

where all  $n_{ij}$  converge absolutely normally and  $f(xyz)$  is a linear function of  $x$ ,  $y$  and  $z$ .

Solving this system, we obtain

$$B_4 = - \frac{(1-\mu_1)(1-\mu_2)(1-\mu_3)}{64} \left\{ n_{22} - \frac{1}{2} [F_1 k_1 L_1 (1-4\mu_4) - F_2 k_2 L_2] - f(xyz) \right\} \cdot \frac{1}{\Delta}$$

$$D_4 = \frac{(1-\mu_1)(1-\mu_2)(1-\mu_3)}{64} [n_{21} - F_1 k_1 L_1] \cdot \frac{1}{\Delta}$$



where

$$\begin{aligned} \nabla = & n_{11} \cdot n_{22} - n_{12} \cdot n_{21} + F_1 k_1 L_1 (n_{12} + n_{22}) - f(xyz) (n_{11} + n_{21}) \\ & - n_{11} \frac{1}{2} [F_1 k_1 L_1 (1 - 4\mu_4) - F_2 k_2 L_2] - n_{21} \frac{1}{2} [F_1 k_1 L_1 (1 - 4\mu_4) + F_2 k_2 L_2] \\ & - F_1 k_1 L_1 \cdot \frac{1}{2} [F_1 k_1 L_1 (1 - 4\mu_4) - F_2 k_2 L_2] + F_1 k_1 L_1 \frac{1}{2} [F_1 k_1 L_1 (1 - 4\mu_4) + F_2 k_2 L_2] \\ & + F_1 k_1 L_1 f(xyz) - F_1 k_1 L_1 f(xyz) \end{aligned}$$

$$\nabla = f(n_{11}, n_{12}, n_{21}, n_{22}) + F_1 k_1 L_1 \cdot F_2 k_2 L_2$$

The linear function  $f(xyz)$  has disappeared in the denominator.

The function  $f(n_{ij})$  converges normally so that the limit value for the denominator is a constant:

$$\lim_{m \rightarrow \infty} \nabla = F_1 k_1 L_1 \cdot F_2 k_2 L_2$$

### 3.2.3. Values of the parameters $A_1, D_1$ .

At the bottom of the first layer, parameters  $A_1$  and  $C_1$  are factors of the positive exponent  $e^x$ . Parameters  $B_1$  and  $D_1$  are factors of the negative exponent  $e^{-x}$ .

Thus when computing stresses in the first layer we have to input at least two positive exponents which necessarily will lead to overflow problems.

Therefore we express in the program next modified values of the parameters  $A_1, B_1, C_1$  and  $D_1$ :  $A_1 \cdot e^x, B_1, C_1 \cdot e^x, D_1$ .

Then if we have to compute a stress, let us say at a depth  $2x/3$ , we multiply the parameters  $A_1 \cdot e^x$  and  $C_1 \cdot e^x$  by the negative exponent  $e^{-x/3}$  and the parameters  $B_1$  and  $D_1$  by the negative exponent  $e^{-2x/3}$  and avoid, in doing so, all overflow problems.

The values of the parameters  $A_1, B_1, C_1$  and  $D_1$  are given by (sol. 3) of §3.2.2. To obtain the values for  $A_1 \cdot e^x, B_1, C_1 \cdot e^x$  and  $D_1$  we split (sol. 3) into two parts:

$$(A_1 \ 0 \ C_1 \ 0)^T + (0 \ B_1 \ 0 \ D_1)^T =$$

$$- \frac{1}{64(1-\mu_1)(1-\mu_2)(1-\mu_3)} \begin{bmatrix} & & & \\ & & & \\ & & & \\ & & & \end{bmatrix} (B_4 \ D_4)^T$$

We notice now that the matrices  $T_{1i}$  and  $T_{4i}$  contain nothing but zeros in their first and third rows, so that

$$T_{1i} \cdot (B_4 \ D_4)^T = \begin{pmatrix} 0 & 0 \\ + & + \\ 0 & 0 \\ + & + \end{pmatrix} (B_4 \ D_4)^T = \begin{pmatrix} 0 \\ + \\ 0 \\ + \end{pmatrix}$$

and that the matrices  $T_{2i}$  and  $T_{3i}$  contain nothing but zeros in their second and fourth rows, so that

$$T_{2i} \cdot (B_4 \ D_4)^T = \begin{pmatrix} + & + \\ 0 & 0 \\ + & + \\ 0 & 0 \end{pmatrix} (B_4 \ D_4)^T = \begin{pmatrix} + \\ 0 \\ + \\ 0 \end{pmatrix}$$

Thus the values of  $A_1$  and  $C_1$  depend only on the matrices  $T_{2i}$  and  $T_{3i}$ . This has to be so, because only the exponents multiplying the matrices  $T_{2i}$  and  $T_{3i}$  do not overflow when they are multiplied by  $e^x$ . The exponents multiplying the matrices  $T_{1i}$  and  $T_{4i}$  could overflow when multiplied by  $e^x$ : for example, the exponent  $e^{-2(t-z)}$  multiplying the matrix  $T_{14}$  if  $x > 2(t-z)$ . We obtain then

$$e^x (A_1 \ 0 \ C_1 \ 0)^T = - \frac{1}{64(1-\mu_1)(1-\mu_2)(1-\mu_3)}.$$

$$\left[ e^{-(2y-x)} e^{-2(t-z)} T_{21} + e^{-(2y-x)} T_{22} + e^{-(2t-x)} T_{23} + e^{-(2z-x)} T_{24} \right. \\ \left. + e^{-x} e^{-2(t-y)} T_{32} + e^{-x} e^{-2(z-y)} T_{33} + e^{-x} e^{-2(t-z)} T_{34} + e^{-x} T_{31} \right] (B_4 \ D_4)^T$$

$$(0 \ B_1 \ 0 \ D_1)^T = - \frac{1}{(1-\mu_1)(1-\mu_2)(1-\mu_3)}.$$

$$\left[ e^{-2(t-y)} T_{12} + e^{-2(z-y)} T_{11} + e^{-2(t-z)} T_{14} + e^{-2(y-x)} e^{-2(t-z)} T_{41} \right. \\ \left. + e^{-2(y-x)} T_{42} + e^{-2(t-x)} T_{43} + e^{-2(z-x)} T_{44} + T_{11} \right] (B_4 \ D_4)^T$$

We follow the same procedure for the parameters  $A_2$ ,  $B_2$ ,  $C_2$  and  $D_2$  and obtain from (sol. 2):

$$e^y (A_2 \ 0 \ C_2 \ 0)^T = \frac{1}{16(1-\mu_2)(1-\mu_3)}.$$

$$\left[ e^{-y} e^{-2(t-z)} N_{21} + e^{-y} N_{22} + e^{-(2t-y)} N_{23} + e^{-(2z-y)} N_{24} \right] (B_4 \ D_4)^T$$

$$(0 \ B_2 \ 0 \ D_2)^T = \frac{1}{16(1-\mu_2)(1-\mu_3)}.$$

$$\left[ e^{-2(t-y)} N_{12} + e^{-2(z-y)} N_{13} + e^{-2(t-z)} N_{14} + N_{11} \right] (B_4 \ D_4)^T$$

Finally we obtain for the parameters  $A_3$ ,  $B_3$ ,  $C_3$  and  $D_3$  (sol. 1):

$$e^z (A_3 \ 0 \ C_3 \ 0)^T = -\frac{1}{4(1-\mu_3)} \left[ e^{-(2k-z)} M_{5161} + e^{-z} M_{5162} \right] (B_4 \ D_4)^T$$

$$(0 \ B_3 \ 0 \ D_3)^T = -\frac{1}{4(1-\mu_3)} \left[ e^{-2(k-z)} M_{5261} + M_{52} \cdot M_{62} \right] (B_4 \ D_4)^T$$

### 3.2.4. The deflection at the surface.

The vertical deflection at the surface is given by

$$W = -pa \int_0^\infty \frac{J_0(mr) \cdot J_1(ma)}{m} \frac{1+\mu_1}{E_1} \left[ A_1 - B_1 - 2C_1(1-2\mu_1) - 2D_1(1-2\mu_1) \right] dm$$

The parameters  $B_1$  and  $D_1$  do not converge normally because they contain a constant in their numerator.

Utilizing the boundary conditions at the surface, we express then  $B_1$  and  $D_1$  in function of  $A_1$  and  $C_1$ :

$$B_1 = 2\mu_1 + A_1(1-4\mu_1) + 4\mu_1 C_1(1-2\mu_1)$$

$$D_1 = 1 - 2A_1 + C_1(1-4\mu_1)$$

so that

$$A_1 - B_1 - 2C_1(1-2\mu_1) - 2D_1(1-2\mu_1) =$$

$$-2(1-\mu_1) [1 - 2A_1 + 2C_1(1-2\mu_1)]$$

and

$$W = \frac{2(1-\mu_1^2)}{E_1} \cdot pa \int_0^\infty \frac{J_0(mr) \cdot J_1(ma)}{m} \cdot dm - \frac{4(1-\mu_1^2)}{E_1} \cdot pa \int_0^\infty \frac{J_0(mr) \cdot J_1(ma)}{m} [A_1 - (1-2\mu_1)C_1] dm$$

The first integral can be solved analytically (§ 2.3). The second one converges absolutely normally.

### 3.2.5. The stresses and displacements in the first layer.

We compute, for example, the vertical stress at a depth  $0 < H < H_1$ .

Its value is given by

$$\sigma_{z,H} = pa \int_0^\infty J_0(mr) \cdot J_1(ma) \left[ A_1 e^x \cdot e^{-(x-mH)} + B_1 e^{-mH} - C_1 e^x (1-2\mu_1 - mH) \cdot e^{-(x-mH)} + D_1 (1-2\mu_1 + mH) \cdot e^{-mH} \right] dm$$

If the value of  $H$  is very small (if we compute a stress near the surface), the values of  $B_1 \cdot e^{-mH}$  and  $D_1 \cdot e^{-mH}$  will converge very slowly.

We know from the preceeding paragraph that

$$B_1 e^{-mH} = 2\mu_1 e^{-mH} + A_1 (1-4\mu_1) e^{-mH} + 4\mu_1 C_1 (1-2\mu_1) e^{-mH}$$

$$D_1 e^{-mH} = e^{-mH} - 2A_1 e^{-mH} + C_1 (1-4\mu_1) e^{-mH}$$

So that

$$B_1 e^{-mH} + D_1 (1-2\mu_1 + mH) e^{-mH} = (1+mH) e^{-mH} - 2A_1 (1+mH) e^{-mH} + C_1 (1-2\mu_1 + mH - 4\mu_1 mH) e^{-mH}$$

We write then

$$\sigma_{z,H} = pa \int_0^\infty J_0(mr) \cdot J_1(ma) \left[ A_1 e^x \cdot e^{-(x-mH)} - C_1 e^x (1-2\mu_1 - mH) \cdot e^{-(x-mH)} + (1+mH) e^{-mH} - 2A_1 (1+mH) e^{-mH} + C_1 (1-2\mu_1 + mH - 4\mu_1 mH) e^{-mH} \right] dm$$

We can again split this integral into

$$\begin{aligned} \sigma_{z,H} = & pa \int_0^\infty J_0(mr) J_1(ma) (1+mH) e^{-mH} dm \\ & + pa \int_0^\infty J_0(mr) \cdot J_1(ma) \left[ A_1 e^x \cdot e^{-(x-mH)} - C_1 e^x (1-2\mu_1 - mH) e^{-(x-mH)} - 2A_1 e^x (1+mH) \cdot e^{-(x+mH)} + C_1 e^x (1-2\mu_1 + mH - 4\mu_1 mH) e^{-(x+mH)} \right] dm \end{aligned}$$

The first integral can be solved analytically (§ 2.4), the second one converges normally.

Of course the same procedure can be applied for the computation of all other stresses or displacements in the first layer.

### 3.2.6. The stresses and displacements in the second layer.

Normally one should not have numerical problems in the computation of stresses and displacements in the second layer, except for the case that the first layer should be very thin (which can happen with overlays).

The terms  $B_2 \cdot e^{-mH}$  and  $D_2 \cdot e^{-mH}$  could then again converge quite slowly. The solution is then obtained as follows.

We write

$$B_{21} = B_2 - \frac{1}{16(1-\mu_2)(1-\mu_3)} \cdot N_{11}(2.1, 2.2) (B_4 \ D_4)^T$$

where  $N_{11}(2.1, 2.2)$  are the first and the second term of the second row of matrix  $N_{11}$ .

$$D_{21} = D_2 - \frac{1}{16(1-\mu_2)(1-\mu_3)} \cdot N_{11}(4.1, 4.2) (B_4 \ D_4)^T$$

$$B_{41} = B_4 - \frac{(1-\mu_1)(1-\mu_2)(1-\mu_3)}{64} \left\{ \frac{1}{2} [F_1 k_1 L_1 (1-4\mu_4) - F_2 k_2 L_2] - i(xyz) \right\} \frac{1}{V}$$

$$= B_4 - B_{42}$$

$$D_{41} = D_4 + \frac{(1-\mu_1)(1-\mu_2)(1-\mu_3)}{64} \cdot F_1 k_1 L_1 \cdot \frac{1}{V}$$

$$= D_4 + D_{42}$$

$$N_{11}(2.1, 2.2) = \begin{pmatrix} k_1 L_1 & k_1 L_3 + k_3 L_2 + \gamma(k_1 - k_2)L_2 + z(L_1 - L_2)k_1 \end{pmatrix}$$

$$N_{11}(4.1, 4.2) = \begin{pmatrix} 0 & k_2 L_2 \end{pmatrix}$$

We split then  $(B_4 \ D_4)^T$  en write

$$B_2 = B_{21} + \frac{1}{16(1-\mu_2)(1-\mu_3)} N_{11}(2.1, 2.2) [(B_{41} \ D_{41})^T + (B_{42} \ D_{42})^T]$$

$$= B_{21} + B_{22} + \frac{1}{16(1-\mu_2)(1-\mu_3)} \cdot N_{11}(2.1, 2.2) (B_{42} \ D_{42})^T$$

$$D_2 = D_{21} + \frac{1}{16(1-\mu_2)(1-\mu_3)} N_{11}(4.1, 4.2) [(B_{41} \ D_{41})^T + (B_{42} \ D_{42})^T]$$

$$= D_{21} + D_{22} + \frac{1}{16(1-\mu_2)(1-\mu_3)} \cdot N_{11}(4.1, 4.2) (B_{42} \ D_{42})^T$$

We notice that the terms  $B_{21}$ ,  $B_{22}$ ,  $D_{21}$  and  $D_{22}$  converge normally.

$$B_{23} = \frac{1}{16(1-\mu_2)(1-\mu_3)} N_{11}(2.1, 2.2) (B_{42} \ D_{42})^T =$$

$$\frac{1-\mu_1}{4} \left\{ \frac{1}{2} [F_1 k_1 L_1 (1-4\mu_4) - F_2 k_2 L_2] + \frac{1}{1} (xyz) \right\} \frac{1}{\nabla}$$

$$- \frac{1-\mu_1}{4} \left\{ k_1 L_3 + L_3 L_2 + y(k_1 - k_2) \cdot L_2 + z(L_1 - L_2) k_1 \right\} \frac{F_1 k_1 L_1}{\nabla}$$

$$= \frac{1-\mu_1}{4} k_1 L_1 k_2 L_2 \left\{ \frac{1}{2} [F_1 (1-4\mu_4) - F_2] + x(F_1 - F_2) \right\} \frac{1}{\nabla}$$

$$D_{23} = \frac{1}{16(1-\mu_2)(1-\mu_3)} N_{11}(4.1, 4.2) (B_{42} \ D_{42})^T =$$

$$- \frac{1-\mu_1}{4} \cdot k_2 L_2 F_1 k_1 L_1 \cdot \frac{1}{\nabla}$$

We split now

$$\nabla = \nabla_1 + F_1 k_1 L_1 \cdot F_2 k_2 L_2$$

where  $\nabla_1$  Converges normally.

We divide then the numerator by the denominator and obtain

$$B_{23} = \frac{1-\mu_1}{4} \cdot \frac{\frac{1}{2} [F_1(1-4\mu_4) - F_2] + x(F_1 - F_2)}{F_1 F_2} \\ - \frac{1-\mu_1}{4} \cdot \frac{\left\{ \frac{1}{2} [F_1(1-4\mu_4) - F_2] + x(F_1 - F_2) \right\} \cdot \nabla_1}{F_1 F_2 \cdot \nabla}$$

The first term can be integrated analytically (§ 2.4), the second one converges normally.

$$D_{23} = - \frac{1-\mu_1}{4} \left[ \frac{1}{F_2} - \frac{\nabla_1}{F_2 \cdot \nabla} \right]$$

### 3.2.7. The stresses and displacements in the third layer.

Following the same procedure as in the preceeding paragraph, we obtain

$$B_{33} = - \frac{(1-\mu_1)(1-\mu_2)}{16} \cdot \frac{\frac{1}{2} [F_1 k_1 (1-4\mu_3) - F_2 k_2] + x(F_1 - F_2) k_2 + y(k_1 - k_2) F_1}{k_1 F_1 \cdot k_2 F_2} \\ + \frac{(1-\mu_1)(1-\mu_2)}{16} \cdot \frac{\left\{ \frac{1}{2} [F_1 k_1 (1-4\mu_3) - F_2 k_2] + x(F_1 - F_2) k_2 + y(k_1 - k_2) F_1 \right\} \cdot \nabla_1}{k_1 F_1 \cdot k_2 F_2 \cdot \nabla}$$

$$D_{33} = \frac{(1-\mu_1)(1-\mu_2)}{16} \left[ \frac{1}{F_2 k_2} - \frac{\nabla_1}{F_2 k_2 \nabla} \right]$$


---



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APPENDIX

Three-layer elastic system

Computer program F77

```

C PROGRAMME DE TRICOUCHE ISOTROPE.
C
C 1ER PARTIE : DECLARATION DES VARIABLES :
C .....
C
      IMPLICIT REAL(A-Z)
      INTEGER J,L,I,Q,M,N,Z7
      CHARACTER*1 AA,BA,ZA*2
      DIMENSION JO(500),J1(500),J2(500),H(9),F(9),Y(9),II(9),JJ(9),E1(1),I(3),N8(3),N9(3),Z1(3),Z2(3),Z3(3),G(9),
      1 ZI(2,4),ZK(4,4),ZL(4,2),ZM11(4,4),ZM12(4,4),ZM21(4,4),ZM22(4,4),ZM31(4,4),ZM32(4,4),ZM41(4,2),ZM42(4,4),
      2 ZAM(4,4),ZBM(4,2),ZCM(4,2),ZAA(2,2),ZBB(2,2),ZCC(2,2),ZDD(2,2),ZAML(4,2),ZBML(4,2),ZM33(4,4),ZM83(4,2)
      DATA(ZI(1,J),I=1,2),J=1,4)/3*1.,-1.,0.,1.,0.,1./
      DATA(ZK(1,J),I=1,4),J=1,4)/16*0./
      DATA(ZL(1,J),I=1,4),J=1,2)/8*0./
      DATA(ZDD(1,J),I=1,2),J=1,2)/4*0./
      DATA(ZM11(1,J),I=1,4),J=1,4)/0.,0.,1.,3*0.,-1.,3*0.,-1.,3*0.,-1.,0.,/
      DATA(ZM12(1,J),I=1,4),J=1,4)/3*0.,-1.,3*0.,-1.,3*0.,-1.,3*0.,-1.,/
      DATA(ZM21(1,J),I=1,4),J=1,4)/1.,1.,14*0.,/
      DATA(ZM22(1,J),I=1,4),J=1,4)/4*0.,1.,-1.,10*0.,/
      DATA(ZM31(1,J),I=1,4),J=1,4)/0.,0.,1.,3*0.,-1.,3*0.,-1.,3*0.,-1.,0.,/
      DATA(ZM32(1,J),I=1,4),J=1,4)/3*0.,-1.,3*0.,-1.,3*0.,-1.,3*0.,-1.,/
      DATA(ZM41(1,J),I=1,4),J=1,2)/1.,-1.,3*0.,1.,0.,0.,/
      DATA(ZM42(1,J),I=1,4),J=1,2)/8*0./
C
      OPEN(1,FILE="RESMIRI190")
C
C 2EME PARTIE : INTRODUCTION DES DONNEES :
C .....
C
      100 WRITE(6,2)
      2 FORMAT("DETERMINATION DES CONTRAINTES ET DEFORMATIONS ",//," DANS UNE STRUCTURE TRICOUCHE ISOTROPE ")
      3 WRITE(6,3)
      3 FORMAT(" A SYMETRIE AXIALE.",//," ET DANS L'AXE DE LA CHARGE.",//)
      998 WRITE(6,998)
      FORMAT(" .....")
      PI=3.141592
      R=0.
      ZA="A1"
      AA="N"
      BA="H"
      11 WRITE(6,11)
      11 FORMAT(" CHOISISSEZ LE PAS D'INTEGRATION (0.1 EST EN GENERAL SUFFISANT) : ")
      READ(5,"(F5.2)")PI
      M1=PI
      M2=20.*M1
      P3=20.
      M1=2.*M1
      M2=M2*10.*M1
      P5=M3*10.
      IF (M2.LT.250.) GOTO 104
C
C 3EME PARTIE : CALCUL DES FONCTIONS DE BESSEL.
C .....
C
      110 M1=PI
      M2=0.
      L=20
      J=0
      P2=M1
      J=J+1

```

```

111 IF (J.GT.M3) GOTO 120
    IF (M.GT.16.) THEN
        J1(J)=SIN(P-PI/4.)*SQRT(2./PI/M)
    ELSE
        Z8=M/2.
        Z9=0.
        V1=1.
        V2=1.
        Z9=Z9+1.
        V2=-V2/Z9+Z8**2/(Z9+1.)
        V1=V1+V2
        IF (ABS(V2).GT..00001) GOTO 111
        J1(J)=Z8+V1
    END IF
    IF (J.LT.L) GOTO 110
    M1=2.*M1
    L=L+10
    GOTO 110

C     4EME PARTIE : INTRUCOCTION DE LA STRUCTURE
C
C .....
120 WRITE(6,20)
20  FORMAT("1 CARACTERISTIQUES GEOMETRIQUES DE LA STRUCTURE",//," EPAISSEUR DE LA COUCHE SUPERIEURE :")
    READ(5,*(F6.3)*M1
21  WRITE(6,21)
    FORMAT("EPAISSEUR DE LA COUCHE INTERMEDIAIRE :")
    READ(5,*(F6.3)*P2
22  WRITE(6,22)
    FORMAT(" RAYON DE LA CHARGE :")
    READ(5,*(F5.2)*RC
23  WRITE(6,23)
24  FORMAT(/," CARACTERISTIQUES MECANQUES DE LA STRUCTURE")
    WRITE(6,27)
25  FORMAT(" MODULE D'ELASTICITE DE LA COUCHE SUPERIEURE ")
    READ(5,*(F12.2)*E1(1)
26  WRITE(6,35)
35  FORMAT(" MODULE D'ELASTICITE DE LA COUCHE INTERMEDIAIRE : ")
    READ(5,*(F12.2)*E2
    IF (E2.NE.E1(1)) GOTO 124
    WRITE(6,36)
36  FORMAT(" ATTENTION :... MODIFIEZ LEGEREMENT CETTE DERNIERE VALEUR")
    WRITE(6,37)
37  FORMAT(" CAR ELLE NE PEUT ETRE EGALE A UN MODULE D'ELASTICITE DE LA COUCHE SUPERIEURE.",//," MERCI.")
    GOTO 122
24  WRITE(6,40)
40  FORMAT(" MODULE ELASTIQUE DU COFFRE : ")
    READ(5,*(F12.2)*E3
    IF (E3.NE.E2) GOTO 125
    WRITE(6,41)
41  FORMAT(" ATTENTION.. MODIFIEZ LEGEREMENT CE COEFFICIENT CAR IL DOIT ETRE DIFFERENT DE CELUI DE LA COUCHE INTERMEDIAIRE ")
    GOTO 124
25  WRITE(6,44)
44  FORMAT("PRESSION APPLIQUEE EN SURFACE : ")
    READ(5,*(F5.2)*P

C     5EME PARTIE : EXECUTION DES CALCULS.
C
C .....
0=1

```

201

```

M120
PUMPI
K=E1(0)/E2
T9=E2/E3
ZK(1,1)=1.+K
ZK(2,2)=1.+K
ZK(3,3)=1.+K
ZK(4,4)=1.+K
ZL(2,1)=1.+T9
ZL(4,2)=1.+T9
ZDD(1,1)=(1.+K)*(1.+T9)
ZDD(2,1)=(1.+K)*(1.+T9)
ZDD(2,2)=(1.+K)*(1.+T9)
M2P0
J=1
L9=20.*P0
L20
DO 211 I=1,9
  Y(I)=0.
  JJ(I)=0.
CONTINUE
CALCUL DE JJ(I) A L'ORIGINE.
Z8=1.
Z9=(1.-K*T9)*(4.*T9+6.*K+2.*K*T9)
Z9=Z9/(-4.*T9+6.*K+3.*K*K-2.*K*K*T9+2.*K*T9-K*K*K*T9+T9)
Z9=Z9*P0/3
JJ(1)=(Z8-Z9)/E1(0)
FIN DU CALCUL DE JJ(1) A L'ORIGINE
CALCUL DES CONSTANTES D'INTEGRATION
X=M*H1/RC
Y9=M*(H1+H2)/RC
X1=EXP(-X)
X2=X1*X1
X3=X1*X2
X4=X2*X2
Y1=EXP(-Y9)
Y2=Y1*Y1
Y3=Y1*Y2
Y4=Y2*Y2
XY=Z8*(Y9-X)
X5=EXP(-XY)
XY1=Z8*Y9-X
Y5=EXP(-XY1)
XY2=Y9-X
X6=EXP(-XY2)
ZM11(1,1)=1.-X
ZM11(1,2)=X
ZM11(1,3)=1.-X
ZM11(1,4)=X
ZM12(2,1)=1.+X
ZM12(2,2)=X
ZM12(2,3)=1.-X
ZM12(2,4)=X
ZM21(3,1)=K

```

```

ZM21(4,1)=K
ZM21(1,3)=X
ZM21(2,3)=1.-X
ZM21(3,3)=K+X
ZM21(4,3)=K+(1.+X)
ZM22(3,2)=K
ZM22(4,2)=K
ZM22(1,4)=X
ZM22(2,4)=1.-X
ZM22(3,4)=K+X
ZM22(4,4)=K+(1.-X)
ZM31(1,1)=1.-Y9
ZM31(1,2)=Y9
ZM31(1,3)=1.-Y9
ZM31(1,4)=Y9
ZM32(2,1)=1.+Y9
ZM32(2,2)=Y9
ZM32(2,3)=1.-Y9
ZM32(2,4)=Y9
ZM41(3,1)=T9
ZM41(4,1)=T9
ZM41(4,2)=T9
ZM42(1,2)=Y9
ZM42(2,2)=Y9
ZM42(3,2)=T9+Y9
ZM42(4,2)=T9+Y9
CALL PRGDUIT2444(ZM11,ZM22,ZAM)
CALL SOMPE42(ZM41,ZM42,ZPES)
CALL PRODUIT4442(ZM31,ZM63,ZCP)
CALL PRGDUIT4444(ZM12,ZM21,ZM33)
CALL PRODUIT4442(ZM33,ZCP,ZBM)
CALL PRODUIT4442(ZAM,ZL,ZAML)
CALL PRODUIT4442(ZK,ZCP,ZKCM)
CALL PRGDUIT2442(ZI,ZAML,ZAA)
CALL PRGDUIT2442(ZI,ZBM,ZBB)
CALL PRGDUIT2442(ZI,ZKCM,ZCC)
UET1=(-ZAA(1,1)+X2+ZBB(1,1)+X5-ZCC(1,1)+Y2)*(-ZAA(2,2)+X2+ZBB(2,2)+X5-ZCC(2,2)+Y2)
UET1=UET1-(-ZAA(1,2)+X2+ZBB(1,2)+X5-ZCC(1,2)+Y2)*(-ZAA(2,1)+X2+ZBB(2,1)+X5-ZCC(2,1)+Y2)
UET1=UET1+ZCC(1,1)*(-ZAA(2,2)+X2+ZBB(2,2)+X5-ZCC(2,2)+Y2)
UET1=UET1+ZCC(2,2)*(-ZAA(1,1)+X2+ZBB(1,1)+X5-ZCC(1,1)+Y2)
UET1=UET1+ZCC(2,1)*(-ZAA(1,2)+X2+ZBB(1,2)+X5-ZCC(1,2)+Y2)
UET1=UET1+ZCC(1,2)*ZDD(2,2)
DET1=UET1+DET2
DET3=4.*(-ZAA(2,2)+X2+ZBB(2,2)+X5-ZCC(2,2)+Y2+ZDD(2,2))/DET
DET3=4.*(-ZAA(2,1)+X2+ZBB(2,1)+X5-ZCC(2,1)+Y2+ZDD(2,1))/DET
UET2=-5*Y1*(ZCP(1,1)+DET3+ZCM(1,2)+DET3)
DET2=-5*(1.+T9)+DET3
DET2=-5*Y1*(ZCP(3,1)+DET3+ZCM(3,2)+DET3)
DET2=5*(1.+T9)+DET3
UET1=-25*(-X1*(ZAML(1,1)+DET3+ZAML(1,2)+DET3)-Y5*(ZKCM(1,1)+DET3+ZKCM(1,2)+DET3))
DET1=-25*(-X1*(ZAML(3,1)+DET3+ZAML(3,2)+DET3)-Y5*(ZKCM(3,1)+DET3+ZKCM(3,2)+DET3))
UET1=-25*(X5*(ZBM(2,1)+DET3+ZBM(2,2)+DET3)+(1.+T9)*(1.+K)+CET3)
DET1=-25*(X5*(ZBM(4,1)+DET3+ZBM(4,2)+DET3)+(1.+T9)*(1.+K)+CET3)

```

FIN DU CALCUL DES CONSTANTES D'INTEGRATION

C  
C  
C

H(1)=2.\*DET1\*X1/Y/E1(G)  
H(2)=DET1+DET1\*X1+DETC1\*X+CETD1\*X\*X1  
H(3)=5\*(DET1+DET1\*X1+(3.+X)\*DETC1-(3.-X)\*DET1\*X1)  
H(4)=DET1+2\*X6+DET1+2\*X1+DETC2\*X\*X6+DET1+2\*X\*X1

Appendix/4

```

M(5)= -5*(DET A2+X6)*DET B2+X1*(3.+X)*DETC2+X6-(3.-X)*DETD2+X1
M(6)=DET A2+DETB2+Y1+DETC2+Y9+DETD2+Y9+Y1
M(7)= -5*(DET A2+DETB2+Y1+(3.+Y9)*DETC2-(3.-Y9)*DETD2+Y1)
M(8)=DETH3+Y1+Y9+DETD3+Y1
M(9)= -5*(DETH3+Y1-DETD3=(3.-Y9)+Y1)
DO 230 I=1,9
    F(I)=M(I)*J(I)
CONTINUE
IF (M1.EQ.0) THEN
    DO 235 I=1,9
        F(I)=4.*F(I)
    CONTINUE
    M1=1
    M=M+P0
    J=J+1
ELSE
    DO 236 I=1,9
        F(I)=2.*F(I)
    CONTINUE
    M1=0
END IF
DO 237 I=1,9
    Y(I)=Y(I)+F(I)
CONTINUE
Z7=1
IF (M1.EQ.1) GOTO 214
E=.000001
DO 238 I=1,9
    IF (ABS(M(I)).GT.E) GOTO 239
CONTINUE
GOTO 245
IF (J.GE.L) THEN
    DO 240 I=1,9
        Y(I)=Y(I)+F(I)/2.
        I1(I)=P0+Y(I)/3.
        JJ(I)=JJ(I)+I1(I)
        Y(I)=0.
    CONTINUE
    P0=2.*P0
    L=L+10
    L9=2.*L9
END IF
M=M+P0
J=J+1
Z7=1
GOTO 214
DO 246 I=1,9
    I1(I)=P0+Y(I)/3.
    JJ(I)=JJ(I)+I1(I)
CONTINUE
DO 895 I=1,9
    C(I)=P+JJ(I)
CONTINUE
C(I)=G(I)+1.5*RC

```

GEPE PARTIE : IMPRESSION DES CONTRAINTES ET DES DEPLACEMENTS  
.....  
WRITE(1,'(///)')

```

70 WRITE(6,70)
   FORMAT("I",//)
   WRITE(1,301)
301 FORMAT(" RESULTATS DANS L'AXE DE LA CHARGE.",/)
   WRITE(6,71)E1(Q),M1,RC,P
71 FORMAT(" E1=",F7.0," U1=0.50 M1=",F5.2," A=",F6.2," P=",F5.2)
   WRITE(6,72)E2,M2,R
72 FORMAT(" E2=",F7.0," U2=0.50 M2=",F5.2," R=",F6.2)
   WRITE(6,73)E3
73 FORMAT(" E3=",F7.0," U3=0.50",//)
   WRITE(6,74)
74 FORMAT("
   WRITE(6,75)
75 FORMAT("
   WRITE(1,76)G(2),G(4),G(6),G(8)
   WRITE(6,76)G(2),G(4),G(6),G(8)
76 FORMAT(" CONTRAINTE VERTICALES * ",F10.6," ",F10.6," ",F10.6)
   WRITE(1,77)G(3),G(5),G(7),G(9)
   WRITE(6,77)G(3),G(5),G(7),G(9)
77 FORMAT(" CONTRAINTE RADIALES * ",F10.6," ",F10.6," ",F10.6)
   WRITE(1,78)G(3),G(5),G(7),G(9)
   WRITE(6,78)G(3),G(5),G(7),G(9)
78 FORMAT(" CONTRAINTE CIRCCONF. * ",F10.6," ",F10.6," ",F10.6)
   WRITE(1,79)G(1)
   WRITE(6,79)G(1)
79 FORMAT(" FLECHES ",F7.4," * * * ")
C
C
C
C
   FIN D'IMPRESSION.
   .....
   WRITE(6,"(" " ",//," LES RESULTATS SONT SUR LE FICHIER RESMTRIISO."
1,"//," C'EST TERME I1",//,"
   END
C
   SUBROUTINE PROCLIT4444(A,B,C)
   DIMENSION A(4,4),B(4,4),C(4,4)
   DO 800 I=1,4
     DO 801 K=1,4
       C(I,K)=0.
     DO 802 J=1,4
       C(I,K)=C(I,K)+A(I,J)*B(J,K)
     CONTINUE
   CONTINUE
   CONTINUE
   END
800
801
802
C
   SUBROUTINE PROCLIT2442(A,B,C)
   DIMENSION A(2,4),B(4,2),C(2,2)
   DO 810 I=1,2
     DO 811 K=1,2
       C(I,K)=0.
     DO 812 J=1,4
       C(I,K)=C(I,K)+A(I,J)*B(J,K)
     CONTINUE
   CONTINUE
   END
810
811
812
813

```



```

810 CONTINUE
811 END
C
SUBROUTINE PROCLIT4442(A,B,C)
DIMENSION A(4,4),B(4,2),C(4,2)
DO 830 I=1,4
DO 831 J=1,2
C(I,J)=0.
DO 832 K=1,4
C(I,K)=C(I,K)+A(I,J)*B(J,K)
CONTINUE
CONTINUE
CONTINUE
END
832
831
830
C
SUBROUTINE SOMME42(A,B,C)
DIMENSION A(4,2),B(4,2),C(4,2)
DO 860 I=1,4
DO 861 J=1,2
C(I,J)=A(I,J)+B(I,J)
CONTINUE
CONTINUE
END
861
860

```

DATE  
L MED  
-8